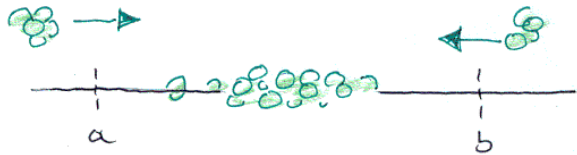


MASS CONSERVATION



RATE OF MASS INCREASE
IN THE INTERVAL

= RATE OF MASS INFLUX
ACROSS THE ENDS

✓ INTERVALS !

$$\frac{\partial}{\partial t} \int_a^b \rho \, dx = \rho_a v_a - \rho_b v_b$$

$$\int_a^b \frac{\partial \rho}{\partial t} \, dx + [\rho v]_a^b = 0$$

$$\int_a^b \left(\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho v) \right) dx = 0$$

✓ $a < b$

IF ρ CONTINUOUS !!

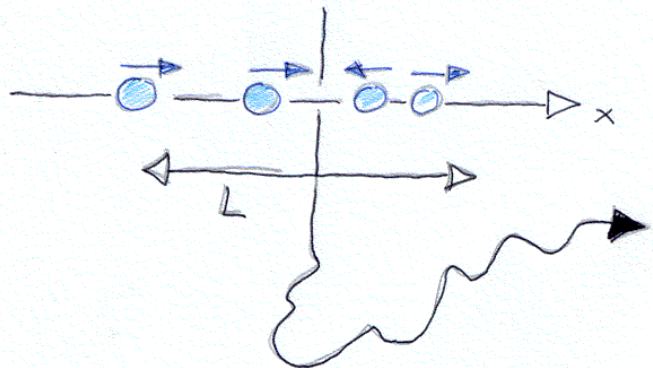
$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho v) = 0$$

CONTINUITY
EQUATION

ALSO

$$\frac{D\rho}{Dt} + \rho \frac{\partial v}{\partial x} = 0$$

MOMENTUM DENSITY

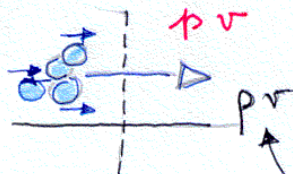


MOMENTUM DENSITY

$$\rho(x, t) \triangleq \underbrace{\rho(x, t)}_{\text{MASS DENSITY}} v(x, t)$$

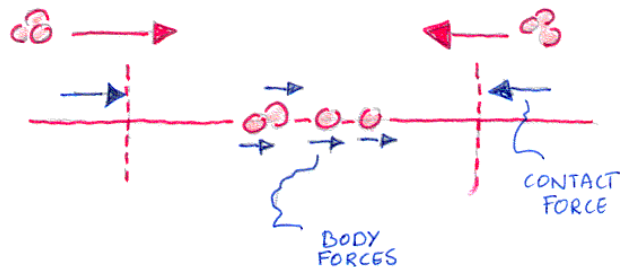
THIS IS IN FACT THE DEFINITION OF THE CONTINUOUS VELOCITY, THE MOMENTUM AND THE MASS DENSITY BEING DEFINED BY THE AVERAGING PROCESS !

MOMENTUM FLUX



MASS FLUX

MOMENTUM CONSERVATION



RATE OF MOMENTUM INCREASE
IN THE INTERVAL

= RATE OF MOMENTUM INFLUX
ACROSS ENDS

+
APPLIED BODY FORCES
+
CONTACT FORCES

✓ INTERVALS !

$$\frac{\partial}{\partial t} \int_a^b \rho \, dx = \rho_A v_A - \rho_B v_B + (-\sigma_A) - (-\sigma_B) + \int_a^b \rho g \, dx$$

σ
 COMPRESSION < 0
 TRACTION > 0

f BODY FORCE
 MASS FORCE EX: GRAVITY

$$\int_a^b \frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho v) \, dx = \int_a^b f + \frac{\partial \sigma}{\partial x} \, dx$$

$\forall a < b$

IF CONTINUITY ASSUMPTIONS !

$$\frac{\partial}{\partial t} (\rho v) + \frac{\partial}{\partial x} (\rho v^2) = f + \frac{\partial \sigma}{\partial x}$$

MOMENTUM EQUATION

..... SOME ALGEBRA !

$$\underbrace{\rho \frac{\partial}{\partial t} v + \rho v \frac{\partial v}{\partial x}}_{\rho \frac{Dv}{Dt}} + \underbrace{v \frac{\partial p}{\partial t} + v \frac{\partial (\rho v)}{\partial x}}_{v \left(\frac{\partial p}{\partial t} + \frac{\partial (\rho v)}{\partial x} \right)} = f + \frac{\partial \sigma}{\partial x}$$

$$\rho \frac{Dv}{Dt}$$

$$v \left(\frac{\partial p}{\partial t} + \frac{\partial (\rho v)}{\partial x} \right)$$

= 0

CONTINUITY
EQUATION

USUAL FORM
OF MOMENTUM EQUATION

$$\rho \frac{Dv}{Dt} = f + \frac{\partial \sigma}{\partial x}$$

NEWTON'S
SECOND
LAW

$$m a = f$$

CONTINUUM PROBLEM

CONTINUITY
EQUATION

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x}(\rho v) = 0$$

MOMENTUM
EQUATION

(NO BODY FORCE $f=0$)

$$\rho \frac{Dv}{Dt} = \frac{\partial \sigma}{\partial x}$$

2

EQUATIONS

3

UNKNOWN

ρ v σ

NEED OF A STATE EQUATION

IDEAL GAS DYNAMICS

$$pV = nRT$$

Diagram labels for $pV = nRT$:

- p : PRESSURE
- V : VOLUME
- n : # MOLES
- R : CONSTANT
- T : TEMPERATURE

$$p = \frac{n}{V}$$

$$\sigma = -p + \underbrace{\text{FRICTION TERMS}}$$

NEGLECTED
HERE

REVERSIBLE
DYNAMICS
IS ONLY CONSIDERED

$$T \propto K p^{\gamma-1}$$

$\gamma = 7/5$ FOR AIR

STATE LAW

$$\begin{aligned} -\sigma &= pRT \\ &= RK p^{\gamma} \\ &= \left(\frac{k^2}{\gamma} \right) p^{\gamma} \end{aligned}$$

k, γ CONSTANTS
OF THE GAS

$$k \triangleq \sqrt{\gamma RK}$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho v) = 0$$

$$\rho \left(\frac{\partial v}{\partial t} + v \frac{\partial v}{\partial x} \right) = -k^2 \rho^{\gamma-1} \frac{\partial \rho}{\partial x}$$

QUASI-LINEAR
PARTIAL
DIFFERENTIAL
EQUATIONS

2 UNKNOWN
2 EQUATIONS

SOLVING THE PROBLEM OF PROPAGATION OF SOUND : DENSITY - VELOCITY WAVES

IDENTIFY
A STATE OF REST

$$\begin{aligned} v &= 0 \\ \rho &= \rho^* \end{aligned}$$

"FIXED
POINT"

THEN, INTRODUCE PERTURBATIONS

$$\begin{aligned} v &= \hat{v} \\ \rho &= \rho^* + \hat{\rho} \end{aligned}$$

"SMALL
PERTURBATIONS"

$$\left\{ \begin{array}{l} \frac{\partial \hat{p}}{\partial t} + p_* \frac{\partial \hat{v}}{\partial x} = 0 \\ (p_* + \hat{p}) \left(\frac{\partial \hat{v}}{\partial t} + \hat{v} \frac{\partial \hat{v}}{\partial x} \right) = -k^2 p_*^{\gamma-1} \left(1 + \frac{\hat{p}}{p_*} \right)^{\gamma-1} \frac{\partial \hat{p}}{\partial x} \end{array} \right.$$

$$\approx p_* \frac{\partial \hat{v}}{\partial t}$$

$$\approx 1$$

$$\begin{aligned} & \frac{\partial \hat{p}}{\partial t} + p_* \frac{\partial \hat{v}}{\partial x} = 0 \\ & p_* \frac{\partial \hat{v}}{\partial t} = -k^2 p_*^{\gamma-1} \frac{\partial \hat{p}}{\partial x} \\ & p_* \frac{\partial^2 \hat{v}}{\partial x \partial t} = -\frac{\partial^2 \hat{p}}{\partial t^2} \\ & p_* \frac{\partial^2 \hat{v}}{\partial x \partial t} = -k^2 p_*^{\gamma-1} \frac{\partial^2 \hat{p}}{\partial x^2} \end{aligned}$$

WAVE
EQUATION

$$\frac{\partial^2 \hat{p}}{\partial t^2} = k^2 p_*^{\gamma-1} \frac{\partial^2 \hat{p}}{\partial x^2}$$

WAVE EQUATION

$$\frac{\partial^2 \hat{p}}{\partial t^2} = \underbrace{k^2 \rho_*^{\gamma-1}}_{\Delta(c^*)^2} \frac{\partial^2 \hat{p}}{\partial x^2}$$

SOLUTION

$$\hat{p}(x, t) = f(x - c^*t) + g(x + c^*t)$$

SPECIFIED
BY BOUNDARY AND
INITIAL CONDITIONS

LOUD UNIDIRECTIONAL SOUND

WITHOUT ANY LOSS OF GENERALITY,

$$\gamma = 1$$

$$f = 0$$

CONSIDER ONLY SOLUTIONS FOR WHICH

$$p = R(r)$$

AND FIND EXACT SOLUTION

$$\left\{ \begin{array}{l} \frac{\partial p}{\partial t} + \frac{\partial}{\partial x}(p r) = 0 \end{array} \right.$$

$$\left\{ \begin{array}{l} p \left(\frac{\partial r}{\partial t} + r \frac{\partial r}{\partial x} \right) = -k^2 \frac{\partial p}{\partial x} \end{array} \right.$$

$$R' \frac{\partial r}{\partial t} + R' \frac{\partial r}{\partial x} r + R \frac{\partial r}{\partial x} = 0$$

$$\left(\frac{\partial r}{\partial t} + r \frac{\partial r}{\partial x} \right) = \frac{-R}{R'} \frac{\partial r}{\partial x}$$

ASSUMPTION $p = R(r)$ IS CONSISTENT ONLY IF THE FACTORS ARE THE SAME!

$$\left(\frac{\partial r}{\partial t} + r \frac{\partial r}{\partial x} \right) = -k^2 \frac{R'}{R} \frac{\partial r}{\partial x}$$

$$(R')^2 k^2 = (R)^2$$

$$\begin{aligned} R &= \exp(\pm r/k + A) \\ &= \underbrace{\exp(A)}_{p^*} \exp(\pm r/k) \end{aligned}$$

$$R' = \pm \frac{p^*}{k} \exp(\pm r/k).$$

$$\frac{\partial r}{\partial t} + (\pm k + r) \frac{\partial r}{\partial x} = 0.$$

+
RIGHT
TRAVELLING
WAVES

-
LEFT
TRAVELLING
WAVES

NON LINEAR
TERM INDUCES
SHOCKS TO FORM
SONIC BOOMS

Que des équations ?

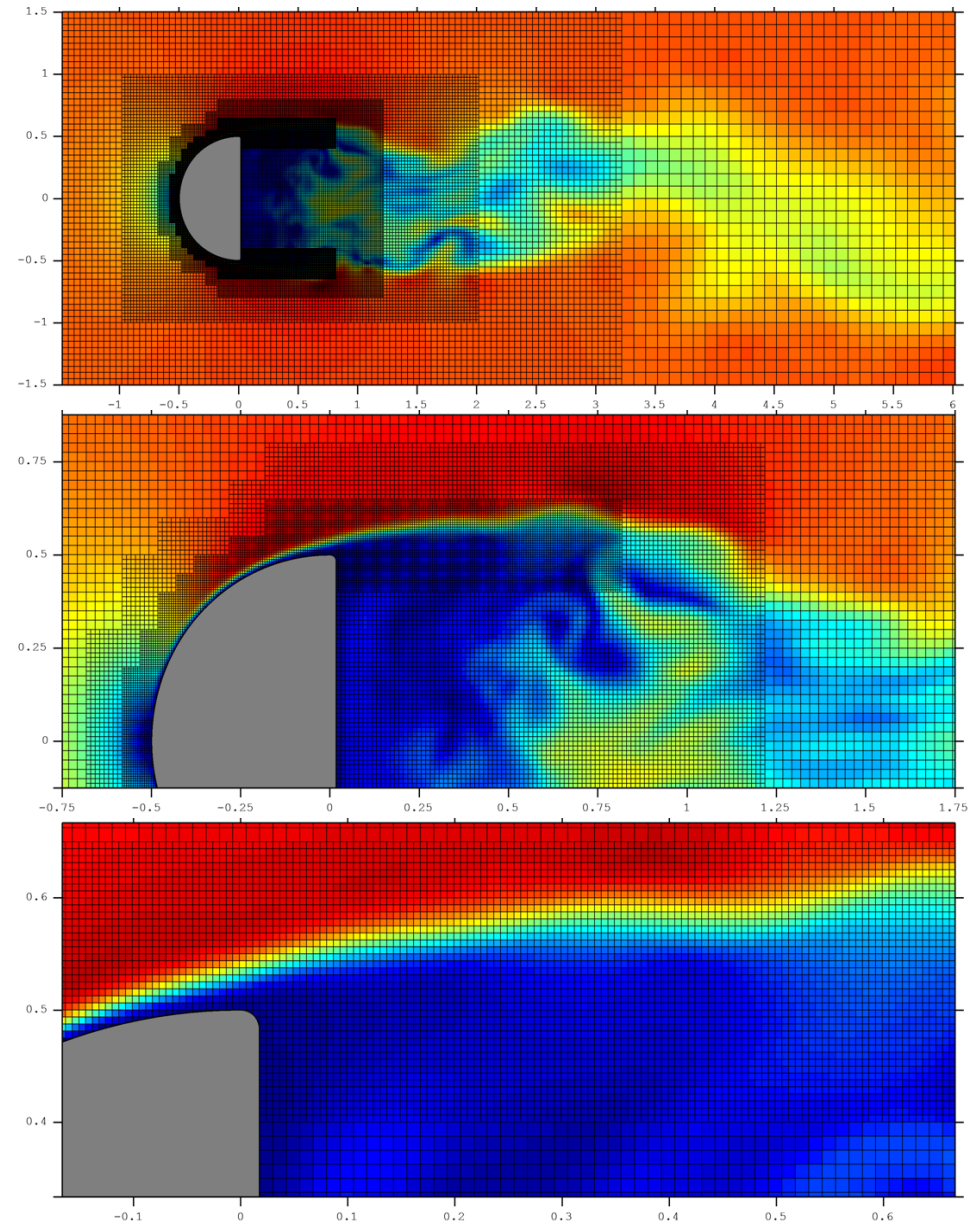
Vortex and Boundary Element Methods

Fast multipole solver

Hierarchically refined grid

Large Eddies Simulation

Flow past a hemisphere at $Re=3000$



Daeninck, Winckelmans, 2006

Nombre de Reynolds

caractérise un écoulement
d'un fluide !

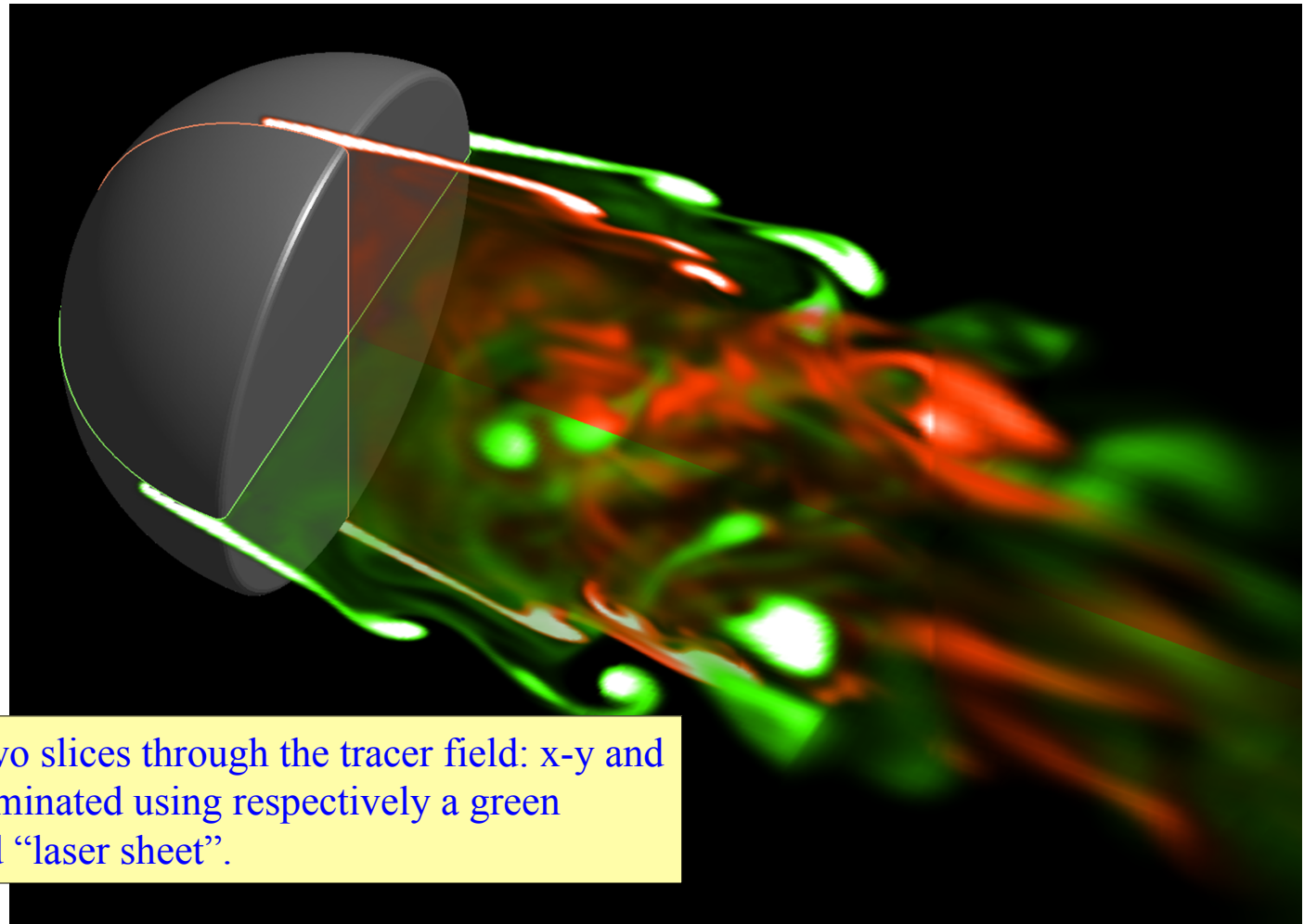
$$Re = \frac{\rho_0 u_0 L}{\mu}$$



Born: 23 Aug 1842 in Belfast, Ireland

Died: 21 Feb 1912 in Watchet, Somerset, England

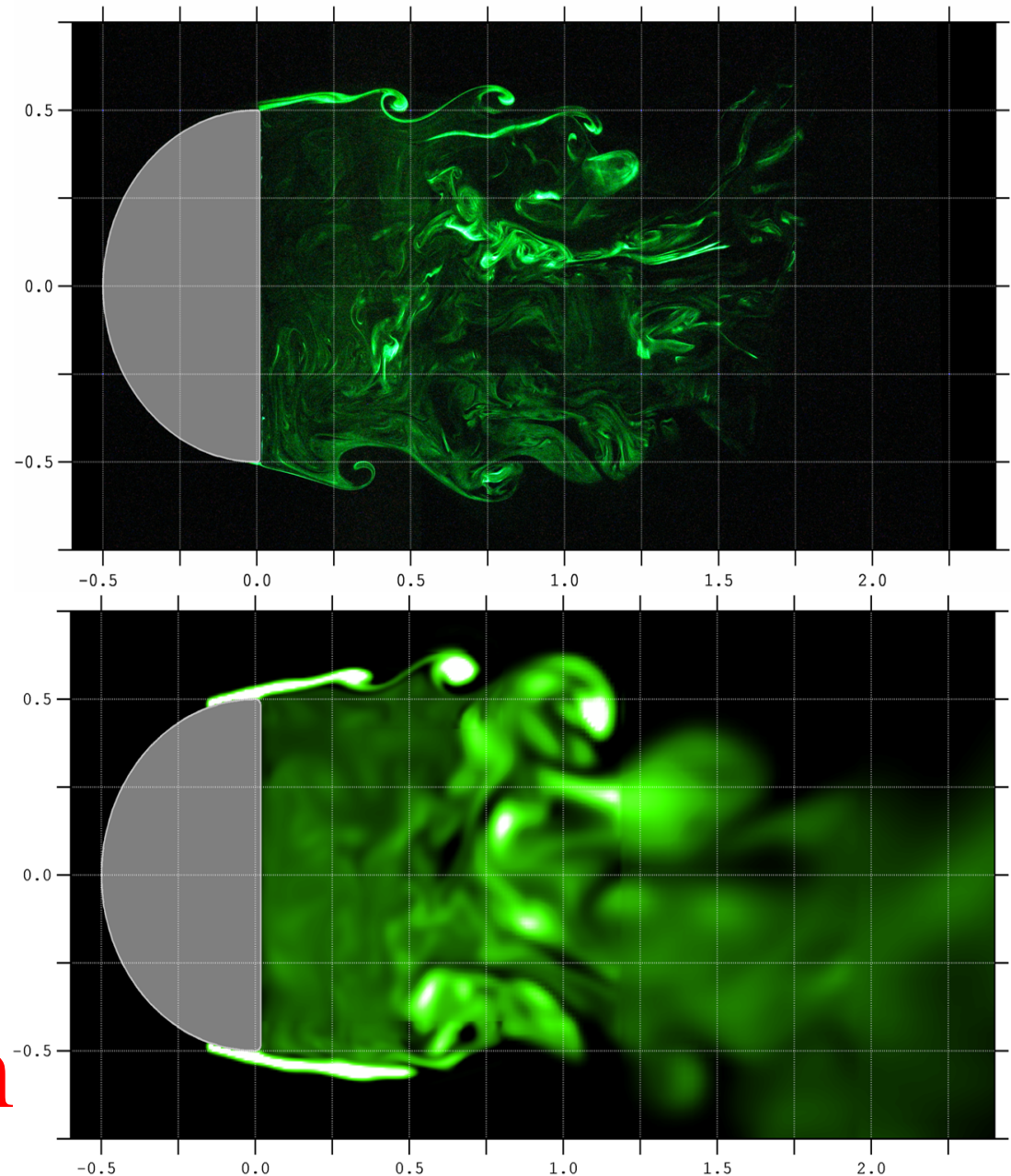
Flow past a hemisphere at $Re=3000$



Perspective view of two slices through the tracer field: x-y and y-z midplanes are illuminated using respectively a green “laser sheet” and a red “laser sheet”.

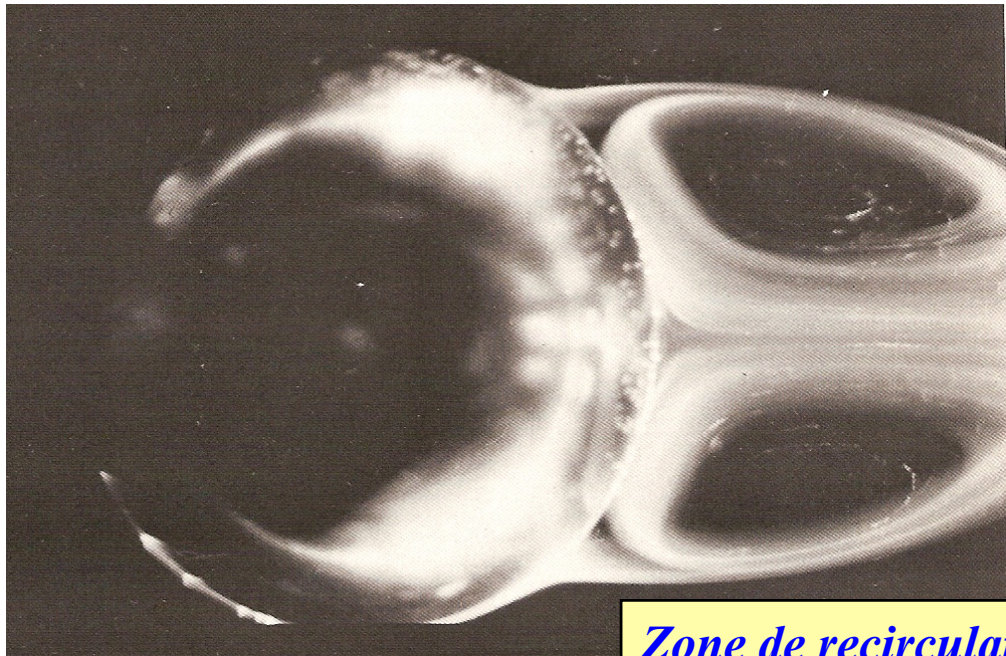
Expérience...

Experimental visualization of the flow past the hemisphere at $Re=3,000$ in a towing tank. Fluorescein is injected in the boundary layer at the front of the hemisphere.



...et simulation

Écoulement laminaire : $Re = 104$



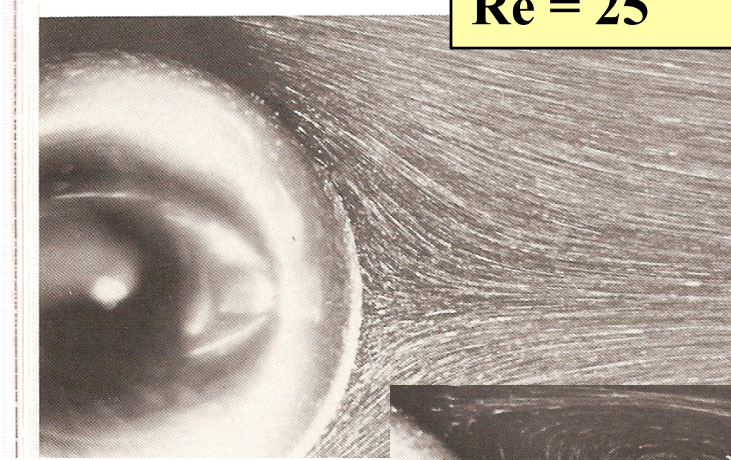
*Zone de recirculation derrière une sphère
 $Re = 104$*

Taneda 1956

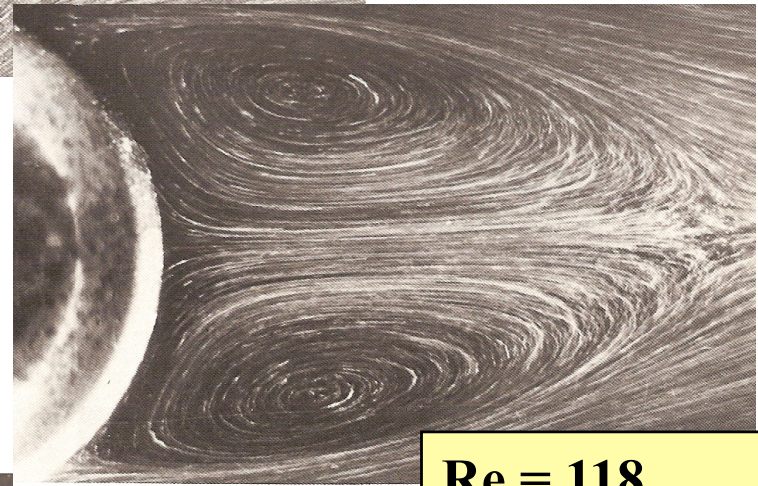
(from An Album of Fluid Motion, Van Dyke)

Écoulement laminaire ?

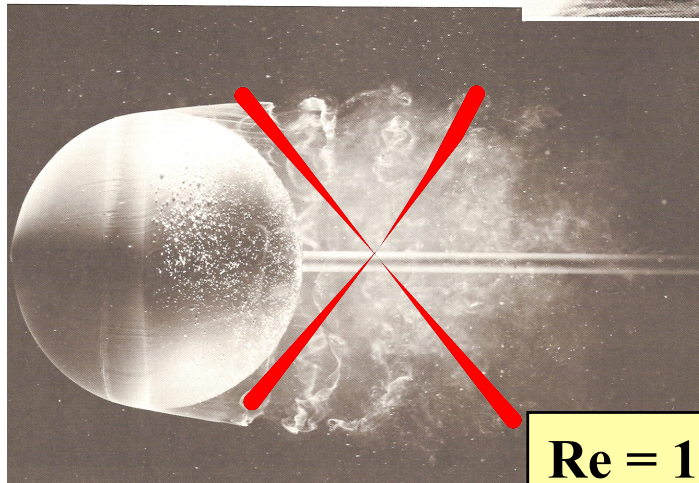
Re = 25



Re = 118



Re = 15000



(Van Dyke, 1982)