

Incompressible turbulent flows: equations for the Reynolds-averaged velocity and temperature fields, and simple cases: channel and boundary layer

Grégoire Winckelmans

NS equations (incompressible)

$$\frac{\partial u_j}{\partial x_j} = 0$$

$$d_{ij} \stackrel{\text{def}}{=} \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

$$\sigma_{ij} \stackrel{\text{def}}{=} -p \delta_{ij} + \tau_{ij}$$

$$\tau_{ij} = 2 \mu d_{ij}$$

$$\begin{aligned} \rho \left(\frac{\partial u_i}{\partial t} + \frac{\partial}{\partial x_j} (u_i u_j) \right) &= \frac{\partial \sigma_{ij}}{\partial x_j} \\ &= -\frac{\partial p}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_j} \\ &= -\frac{\partial p}{\partial x_i} + \mu \frac{\partial}{\partial x_j} \frac{\partial u_i}{\partial x_j} \\ \rho \left(\frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} \right) &= -\frac{\partial p}{\partial x_i} + \mu \frac{\partial}{\partial x_j} \frac{\partial u_i}{\partial x_j} \end{aligned}$$

Reynolds averaging

$$\overline{\phi}(t) \stackrel{\text{def}}{=} \frac{1}{T} \int_{t-T/2}^{t+T/2} \phi(\tau) d\tau$$

**time window used
for the averaging**
 $T_f \ll T \ll T_v$

$$\phi = \overline{\phi} + \phi'$$

**fast time scale of the
turbulent fluctuations**

**slow time scale
of interest**

$$\psi = \overline{\psi} + \psi'$$

$$\overline{\phi \psi} = \overline{\phi} \overline{\psi} + \overline{\phi' \psi'}$$

Reynolds averaged NS (RANS) equations

$$u_i = \bar{u}_i + u'_i$$

$$p = \bar{p} + p'$$

$$u_j = \bar{u}_j + u'_j$$

$$d_{ij} = \bar{d}_{ij} + d'_{ij}$$

$$\tau_{ij} = \bar{\tau}_{ij} + \tau'_{ij}$$

$$\bar{\tau}_{ij} = 2 \mu \bar{d}_{ij}$$



$$\overline{u_i u_j} = \bar{u}_i \bar{u}_j + \overline{u'_i u'_j}$$

$$\bar{\sigma}_{ij} = -\bar{p} \delta_{ij} + \bar{\tau}_{ij}$$



$$\frac{\partial \bar{u}_j}{\partial x_j} = 0$$

$$\rho \left(\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial}{\partial x_j} (\bar{u}_i \bar{u}_j) \right) = -\frac{\partial \bar{p}}{\partial x_i} + \frac{\partial \bar{\tau}_{ij}}{\partial x_j} - \rho \frac{\partial}{\partial x_j} \left(\overline{u'_i u'_j} \right)$$

Reynolds averaged NS (RANS) equations

Reynolds stress : $\bar{\sigma}_{ij}^t \stackrel{\text{def}}{=} -\rho \overline{u'_i u'_j}$

turbulent kinetic energy (TKE) : $\bar{k} \stackrel{\text{def}}{=} \frac{\overline{u'_k u'_k}}{2}$

 $\bar{\sigma}_{kk}^t = -\rho \overline{u'_k u'_k} = -2 \rho \bar{k}$

turbulent shear stress : $\bar{\tau}_{ij}^t \stackrel{\text{def}}{=} \bar{\sigma}_{ij}^t - \frac{1}{3} \bar{\sigma}_{kk}^t \delta_{ij} = \bar{\sigma}_{ij}^t + \frac{2}{3} \rho \bar{k} \delta_{ij}$

 $\rho \left(\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial}{\partial x_j} (\bar{u}_i \bar{u}_j) \right) = \frac{\partial}{\partial x_j} (\bar{\sigma}_{ij} + \bar{\sigma}_{ij}^t)$

$$\rho \left(\frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} \right) = -\frac{\partial}{\partial x_i} \left(\bar{p} + \frac{2}{3} \rho \bar{k} \right) + \frac{\partial}{\partial x_j} (\bar{\tau}_{ij} + \bar{\tau}_{ij}^t)$$

Effective « turbulent viscosity » model

$$\rho \left(\frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} \right) = - \frac{\partial}{\partial x_i} \left(\bar{p} + \frac{2}{3} \rho \bar{k} \right) + \frac{\partial}{\partial x_j} (\bar{\tau}_{ij} + \bar{\tau}_{ij}^t)$$

model the turbulent shear stress using an effective « turbulent viscosity » :

$$\bar{\tau}_{ij}^t \stackrel{\text{model}}{=} 2\mu_t \bar{d}_{ij}$$

effective pressure: $\bar{p}_{\text{eff}} \stackrel{\text{def}}{=} \bar{p} + \frac{2}{3} \rho \bar{k}$


$$\rho \left(\frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} \right) = - \frac{\partial \bar{p}_{\text{eff}}}{\partial x_i} + \frac{\partial}{\partial x_j} (2(\mu + \mu_t) \bar{d}_{ij})$$

We still need to model μ_t !

Momentum equation divided by the density

kinematic viscosity : $\nu \stackrel{\text{def}}{=} \frac{\mu}{\rho}$

turbulent kinematic viscosity : $\nu_t \stackrel{\text{def}}{=} \frac{\mu_t}{\rho}$

kinematic pressure : $\bar{P} \stackrel{\text{def}}{=} \frac{\bar{p}}{\rho}$

effective kinematic pressure : $\bar{P}_{\text{eff}} \stackrel{\text{def}}{=} \bar{P} + \frac{2}{3} \bar{k}$



$$\frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} = - \frac{\partial \bar{P}_{\text{eff}}}{\partial x_i} + \frac{\partial}{\partial x_j} (2 (\nu + \nu_t) \bar{d}_{ij})$$

We still need to model ν_t !

Temperature equation (incompressible)

$$q_j = -k \frac{\partial T}{\partial x_j}$$

$$\begin{aligned} \rho c \left(\frac{\partial T}{\partial t} + \frac{\partial}{\partial x_j} (T u_j) \right) &= \tau_{ij} d_{ij} - \frac{\partial q_j}{\partial x_j} \\ &= 2\mu d_{ij} d_{ij} + k \frac{\partial}{\partial x_j} \frac{\partial T}{\partial x_j} \\ \rho c \left(\frac{\partial T}{\partial t} + u_j \frac{\partial T}{\partial x_j} \right) &= 2\mu d_{ij} d_{ij} + k \frac{\partial}{\partial x_j} \frac{\partial T}{\partial x_j} \end{aligned}$$

Reynolds averaged temperature equation

$$T = \bar{T} + T'$$

$$u_j = \bar{u}_j + u'_j$$

$$d_{ij} = \bar{d}_{ij} + d'_{ij}$$

$$\overline{T u_j} = \bar{T} \bar{u}_j + \overline{T' u'_j}$$

$$\overline{d_{ij} d_{ij}} = \bar{d}_{ij} \bar{d}_{ij} + \overline{d'_{ij} d'_{ij}}$$

$$\rho c \left(\frac{\partial \bar{T}}{\partial t} + \frac{\partial}{\partial x_j} (\bar{T} \bar{u}_j) \right) = 2\mu \left(\bar{d}_{ij} \bar{d}_{ij} + \overline{d'_{ij} d'_{ij}} \right) - \frac{\partial \bar{q}_j}{\partial x_j} - \rho c \frac{\partial}{\partial x_j} (\overline{T' u'_j})$$

turbulent heat flux density : $\bar{q}_j^t \stackrel{\text{def}}{=} \rho c \overline{T' u'_j}$

rate of dissipation of the TKE : $\rho \bar{\epsilon} \stackrel{\text{def}}{=} 2\mu \overline{d'_{ij} d'_{ij}}$

$$\rho c \left(\frac{\partial \bar{T}}{\partial t} + \bar{u}_j \frac{\partial \bar{T}}{\partial x_j} \right) = 2\mu \bar{d}_{ij} \bar{d}_{ij} + \rho \bar{\epsilon} - \frac{\partial}{\partial x_j} (\bar{q}_j + \bar{q}_j^t)$$

Effective « turbulent thermal conductivity » model

$$\rho c \left(\frac{\partial \bar{T}}{\partial t} + \bar{u}_j \frac{\partial \bar{T}}{\partial x_j} \right) = 2\mu \bar{d}_{ij} \bar{d}_{ij} + \rho \bar{\epsilon} - \frac{\partial}{\partial x_j} (\bar{q}_j + \bar{q}_j^t)$$

model the turbulent heat flux density using an effective « turbulent thermal conductivity » :

$$\bar{q}_j^t \stackrel{\text{model}}{=} -k_t \frac{\partial \bar{T}}{\partial x_j}$$

turbulence « at equilibrium » : rate of dissipation of TKE = rate of production of TKE
(see the TKE equation; not developed in this course) :

$$\rho \bar{\epsilon} = \bar{\tau}_{ij}^t \bar{d}_{ij} = 2\mu_t \bar{d}_{ij} \bar{d}_{ij}$$



$$\rho c \left(\frac{\partial \bar{T}}{\partial t} + \bar{u}_j \frac{\partial \bar{T}}{\partial x_j} \right) = 2(\mu + \mu_t) \bar{d}_{ij} \bar{d}_{ij} + \frac{\partial}{\partial x_j} \left((k + k_t) \frac{\partial \bar{T}}{\partial x_j} \right)$$

turbulent Prandtl number : $Pr_t \stackrel{\text{def}}{=} \frac{\mu_t c}{k_t}$ **assumed constant !**

Temperature equation divided by the density and the specific heat

thermal diffusivity : $\alpha \stackrel{\text{def}}{=} \frac{k}{\rho c}$

turbulent thermal diffusivity : $\alpha_t \stackrel{\text{def}}{=} \frac{k_t}{\rho c}$


$$\frac{\partial \bar{T}}{\partial t} + \bar{u}_j \frac{\partial \bar{T}}{\partial x_j} = \frac{2}{c} (\nu + \nu_t) \bar{d}_{ij} \bar{d}_{ij} + \frac{\partial}{\partial x_j} \left((\alpha + \alpha_t) \frac{\partial \bar{T}}{\partial x_j} \right)$$

Prandtl number : $Pr \stackrel{\text{def}}{=} \frac{\mu c}{k} = \frac{\nu}{\alpha}$



turbulent Prandtl number : $Pr_t \stackrel{\text{def}}{=} \frac{\mu_t c}{k_t} = \frac{\nu_t}{\alpha_t}$ **assumed constant !**

$$\frac{\partial \bar{T}}{\partial t} + \bar{u}_j \frac{\partial \bar{T}}{\partial x_j} = \frac{2}{c} (\nu + \nu_t) \bar{d}_{ij} \bar{d}_{ij} + \frac{\partial}{\partial x_j} \left(\left(\frac{\nu}{Pr} + \frac{\nu_t}{Pr_t} \right) \frac{\partial \bar{T}}{\partial x_j} \right)$$

Steady and 2-D mean flows

$$\frac{\partial \bar{u}}{\partial x} + \frac{\partial \bar{v}}{\partial y} = 0$$

$$\frac{\partial}{\partial x} (\bar{u} \bar{u} + \overline{u' u'}) + \frac{\partial}{\partial y} (\bar{u} \bar{v} + \overline{u' v'}) = -\frac{\partial \bar{P}}{\partial x} + \nu \left(\frac{\partial^2 \bar{u}}{\partial x^2} + \frac{\partial^2 \bar{u}}{\partial y^2} \right)$$

$$\frac{\partial}{\partial x} (\bar{v} \bar{u} + \overline{v' u'}) + \frac{\partial}{\partial y} (\bar{v} \bar{v} + \overline{v' v'}) = -\frac{\partial \bar{P}}{\partial y} + \nu \left(\frac{\partial^2 \bar{v}}{\partial x^2} + \frac{\partial^2 \bar{v}}{\partial y^2} \right)$$



$$\bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} = -\frac{\partial \bar{P}}{\partial x} + \frac{\partial}{\partial x} \left(\nu \frac{\partial \bar{u}}{\partial x} - \overline{u' u'} \right) + \frac{\partial}{\partial y} \left(\nu \frac{\partial \bar{u}}{\partial y} - \overline{u' v'} \right)$$

$$\bar{u} \frac{\partial \bar{v}}{\partial x} + \bar{v} \frac{\partial \bar{v}}{\partial y} = -\frac{\partial \bar{P}}{\partial y} + \frac{\partial}{\partial x} \left(\nu \frac{\partial \bar{v}}{\partial x} - \overline{u' v'} \right) + \frac{\partial}{\partial y} \left(\nu \frac{\partial \bar{v}}{\partial y} - \overline{v' v'} \right)$$

Steady and 2-D mean flows: model

$$\frac{\overline{\tau}_{xx}^t}{\rho} = \frac{\overline{\sigma}_{xx}^t}{\rho} + \frac{2}{3} \overline{k} = -\overline{u'u'} + \frac{2}{3} \overline{k} \stackrel{\text{model}}{=} 2\nu_t \frac{\partial \overline{u}}{\partial x}$$

$$\frac{\overline{\tau}_{yy}^t}{\rho} = \frac{\overline{\sigma}_{yy}^t}{\rho} + \frac{2}{3} \overline{k} = -\overline{v'v'} + \frac{2}{3} \overline{k} \stackrel{\text{model}}{=} 2\nu_t \frac{\partial \overline{v}}{\partial y}$$

$$\frac{\overline{\tau}_{xy}^t}{\rho} = \frac{\overline{\sigma}_{xy}^t}{\rho} = -\overline{u'v'} \stackrel{\text{model}}{=} \nu_t \left(\frac{\partial \overline{u}}{\partial y} + \frac{\partial \overline{v}}{\partial x} \right)$$



$$\overline{u} \frac{\partial \overline{u}}{\partial x} + \overline{v} \frac{\partial \overline{u}}{\partial y} = -\frac{\partial \overline{P}_{\text{eff}}}{\partial x} + \frac{\partial}{\partial x} \left(2(\nu + \nu_t) \frac{\partial \overline{u}}{\partial x} \right) + \frac{\partial}{\partial y} \left((\nu + \nu_t) \left(\frac{\partial \overline{u}}{\partial y} + \frac{\partial \overline{v}}{\partial x} \right) \right)$$

$$\overline{u} \frac{\partial \overline{v}}{\partial x} + \overline{v} \frac{\partial \overline{v}}{\partial y} = -\frac{\partial \overline{P}_{\text{eff}}}{\partial y} + \frac{\partial}{\partial x} \left((\nu + \nu_t) \left(\frac{\partial \overline{v}}{\partial x} + \frac{\partial \overline{u}}{\partial y} \right) \right) + \frac{\partial}{\partial y} \left(2(\nu + \nu_t) \frac{\partial \overline{v}}{\partial y} \right)$$

Steady and 2-D mean flows: model

$$\frac{\partial}{\partial x} (\bar{T} \bar{u} + \overline{T' u'}) + \frac{\partial}{\partial y} (\bar{T} \bar{v} + \overline{T' v'}) = \frac{2}{c} \nu (\bar{d}_{ij} \bar{d}_{ij} + \overline{d'_{ij} d'_{ij}}) + \alpha \left(\frac{\partial^2 \bar{T}}{\partial x^2} + \frac{\partial^2 \bar{T}}{\partial y^2} \right)$$



$$\bar{u} \frac{\partial \bar{T}}{\partial x} + \bar{v} \frac{\partial \bar{T}}{\partial y} = \frac{2}{c} \nu (\bar{d}_{ij} \bar{d}_{ij} + \overline{d'_{ij} d'_{ij}}) + \frac{\partial}{\partial x} \left(\alpha \frac{\partial \bar{T}}{\partial x} - \overline{T' u'} \right) + \frac{\partial}{\partial y} \left(\alpha \frac{\partial \bar{T}}{\partial y} - \overline{T' v'} \right)$$



model

$$\bar{u} \frac{\partial \bar{T}}{\partial x} + \bar{v} \frac{\partial \bar{T}}{\partial y} = \frac{2}{c} (\nu + \nu_t) \bar{d}_{ij} \bar{d}_{ij} + \frac{\partial}{\partial x} \left((\alpha + \alpha_t) \frac{\partial \bar{T}}{\partial x} \right) + \frac{\partial}{\partial y} \left((\alpha + \alpha_t) \frac{\partial \bar{T}}{\partial y} \right)$$

with $\bar{d}_{ij} \bar{d}_{ij} = \left(\frac{\partial \bar{u}}{\partial x} \right)^2 + 2 \left(\frac{1}{2} \left(\frac{\partial \bar{u}}{\partial y} + \frac{\partial \bar{v}}{\partial x} \right) \right)^2 + \left(\frac{\partial \bar{v}}{\partial y} \right)^2$

$$\alpha = \frac{\nu}{Pr} \quad \alpha_t = \frac{\nu_t}{Pr_t}$$

Fully developed turbulent channel flow

$$\overline{v} = 0$$

$\overline{u}$, $\overline{u'u'}$, $\overline{v'v'}$, $\overline{w'w'}$, $\overline{k}$, $\overline{u'v'}$ are solely functions of y


$$0 = -\frac{\partial \overline{P}}{\partial x} + \frac{d}{dy} \left(\nu \frac{d\overline{u}}{dy} - \overline{u'v'} \right) \quad \text{also : } \frac{\partial \overline{P}}{\partial x} = \frac{\partial \overline{P}_{\text{eff}}}{\partial x}$$

$$0 = -\frac{\partial \overline{P}}{\partial y} - \frac{d}{dy} (\overline{v'v'}) \quad \Rightarrow \quad \overline{P} + \overline{v'v'} = \overline{P}_w(x)$$


$$\frac{\partial \overline{P}}{\partial x} = \frac{d\overline{P}_w}{dx}$$


$$\frac{d}{dy} \left(\nu \frac{d\overline{u}}{dy} - \overline{u'v'} \right) = \frac{d\overline{P}_w}{dx} = \text{Constant}$$

Fully developed turbulent channel flow

$$\frac{d}{dy} \left(\nu \frac{d\bar{u}}{dy} - \overline{u'v'} \right) = \frac{d\bar{P}_w}{dx} = \text{Constant}$$



Note: y is here measured from the bottom wall

$$\nu \frac{d\bar{u}}{dy} - \overline{u'v'} = -\frac{d\bar{P}_w}{dx} (h - y) = -\frac{d\bar{P}_w}{dx} h \left(1 - \frac{y}{h} \right)$$

$$\mu \frac{d\bar{u}}{dy} - \rho \overline{u'v'} = -\frac{d\bar{p}_w}{dx} h \left(1 - \frac{y}{h} \right)$$

$$\bar{\tau}_{xy} + \bar{\tau}_{xy}^t = -\frac{d\bar{p}_w}{dx} h \left(1 - \frac{y}{h} \right)$$

The profile of the total shear stress (molecular + turbulent) is linear !

Fully developed turbulent channel flow

$$\bar{\tau}_{xy}(y) + \bar{\tau}_{xy}^t(y) = -\frac{d\bar{p}_w}{dx} h \left(1 - \frac{y}{h}\right)$$



short notation : $\bar{\tau}(y) = \bar{\tau}_{xy}(y)$

$$\bar{\tau}^t(y) = \bar{\tau}_{xy}^t(y)$$

$$\bar{\tau}_w = -\frac{d\bar{p}_w}{dx} h$$

$$\bar{\tau}(y) + \bar{\tau}^t(y) = \bar{\tau}_w \left(1 - \frac{y}{h}\right)$$



friction velocity : $\bar{u}_\tau \stackrel{\text{def}}{=} \sqrt{\frac{\bar{\tau}_w}{\rho}}$

$$\nu \frac{d\bar{u}}{dy}(y) - \overline{u'v'}(y) = \bar{u}_\tau^2 \left(1 - \frac{y}{h}\right)$$

$$\nu \frac{d\bar{u}}{dy}(y) + \nu_t(y) \frac{d\bar{u}}{dy}(y) = \bar{u}_\tau^2 \left(1 - \frac{y}{h}\right)$$

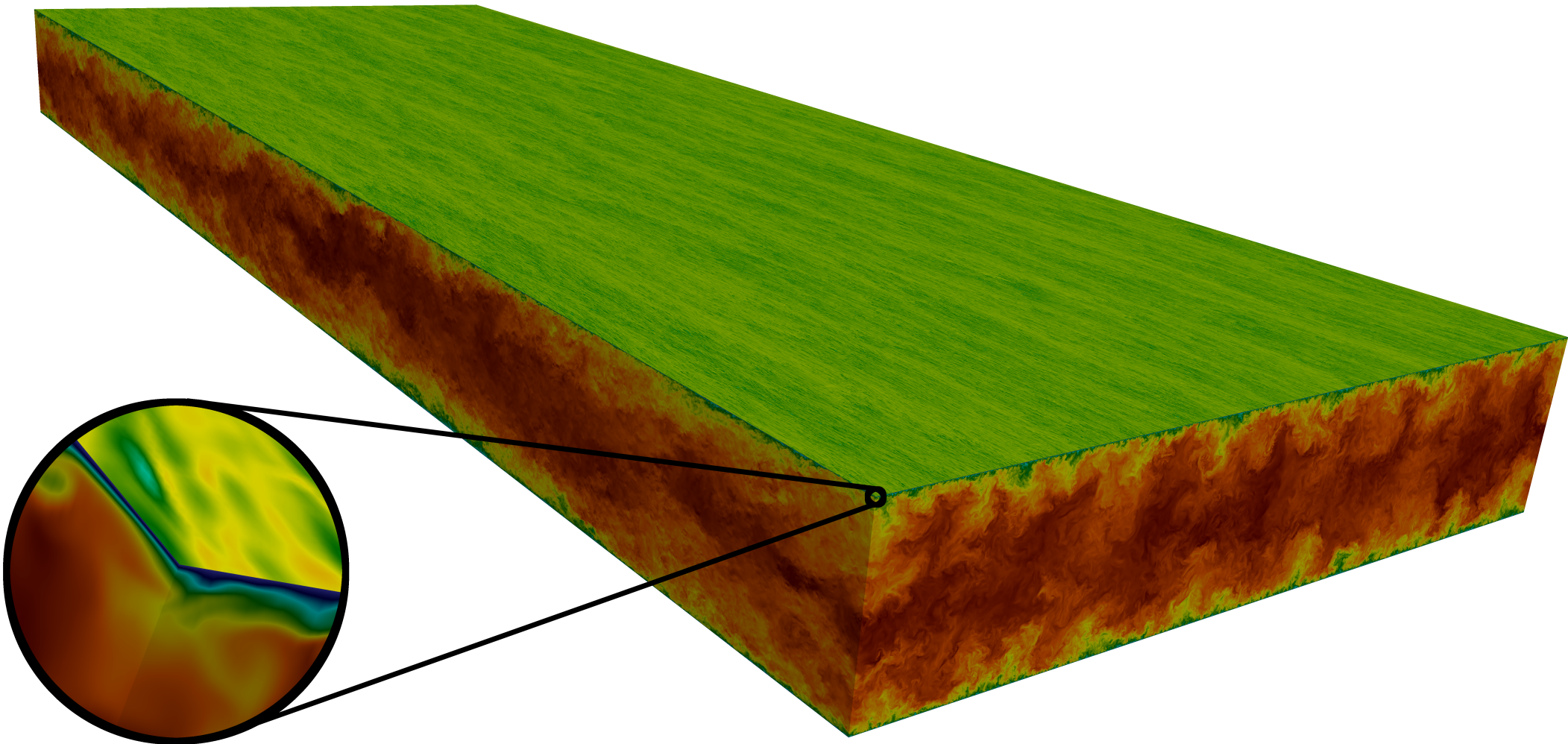
High Re channel flow simulation

$$Re_\tau \stackrel{\text{def}}{=} \frac{h \bar{u}_\tau}{\nu} = 5200$$

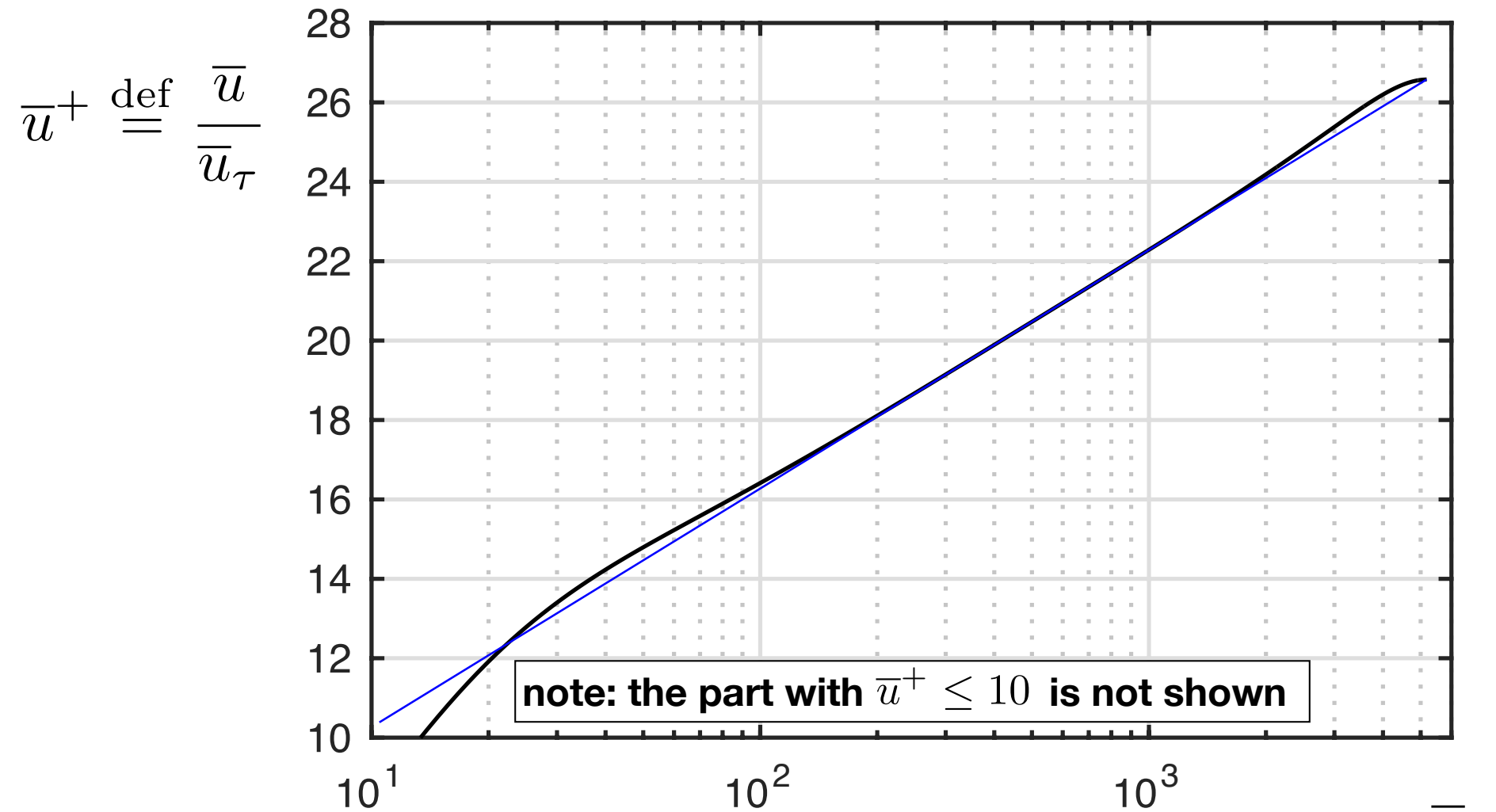
Lee and Moser 2015

DNS = Direct Numerical Simulation

all the scales are resolved !



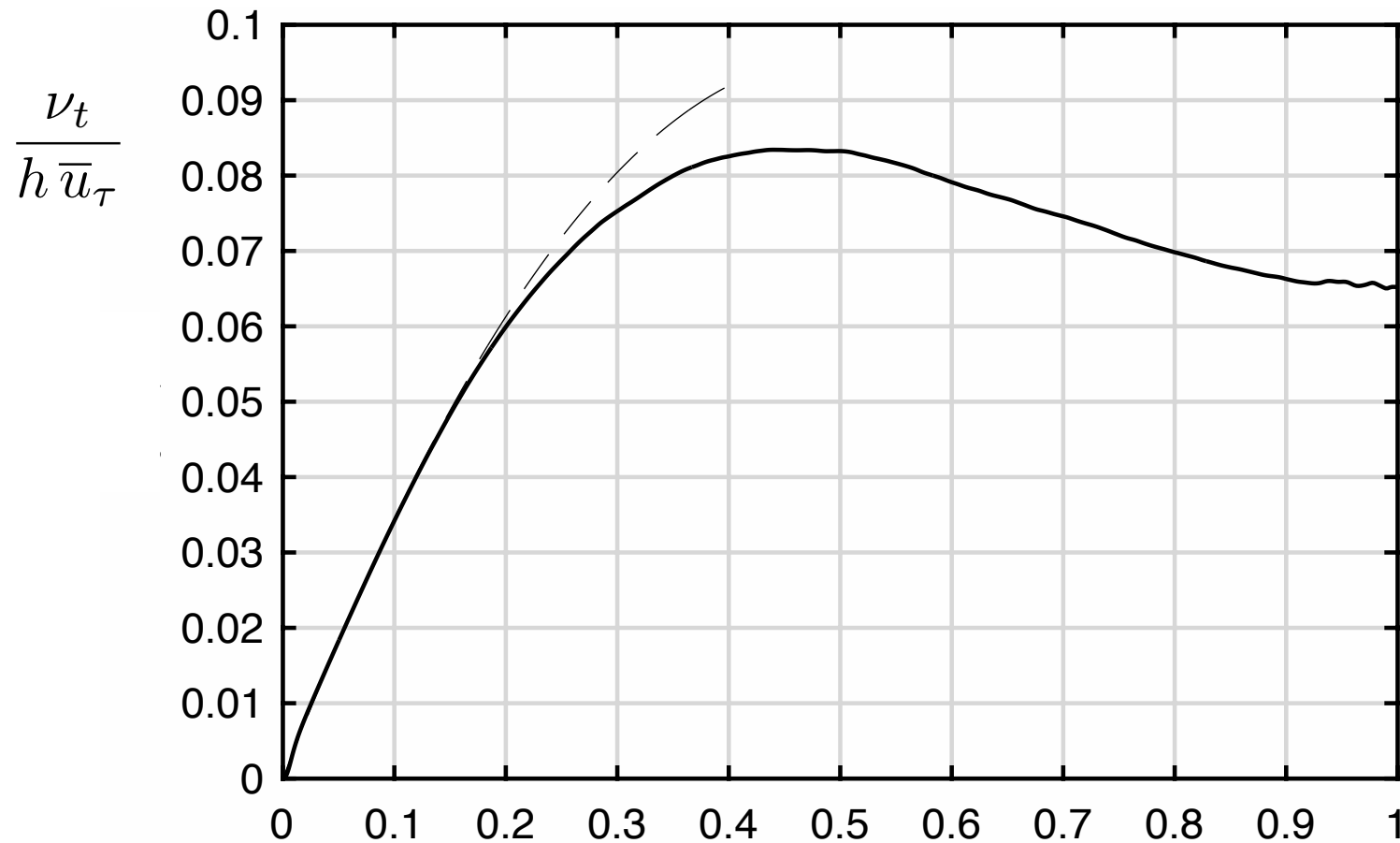
velocity profile obtained from the DNS



log law (in blue) : $\bar{u}^+ = \frac{1}{\kappa} \log y^+ + C$

$$y^+ \stackrel{\text{def}}{=} \frac{y \bar{u}_\tau}{\nu}$$

turbulent viscosity obtained from the DNS

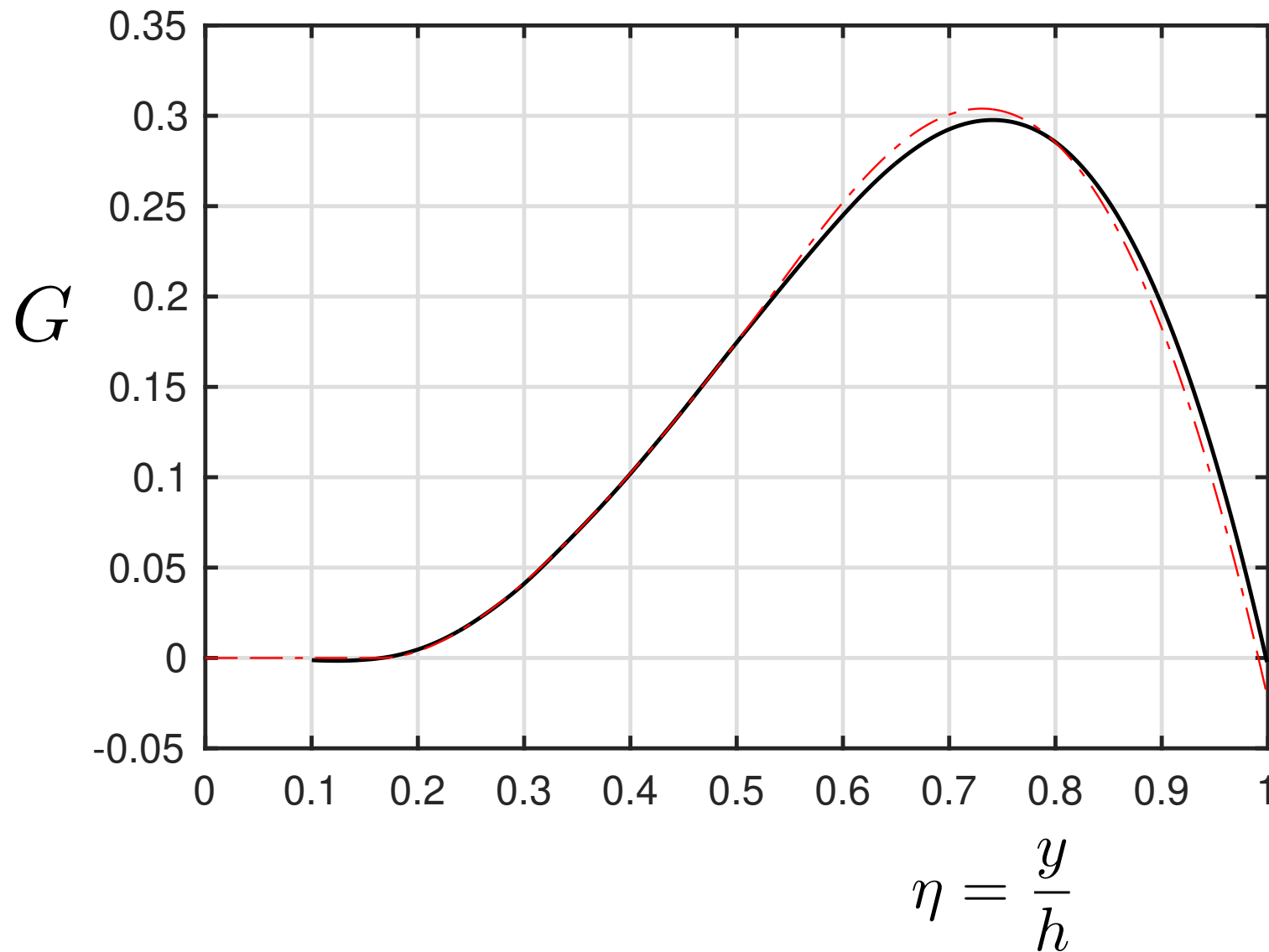


behaviour in the near wall log layer (in dash):

$$\frac{\nu_t}{h \bar{u}_\tau} = \kappa \frac{y}{h} \left(1 - \frac{y}{h} \right)$$

$$\eta = \frac{y}{h}$$

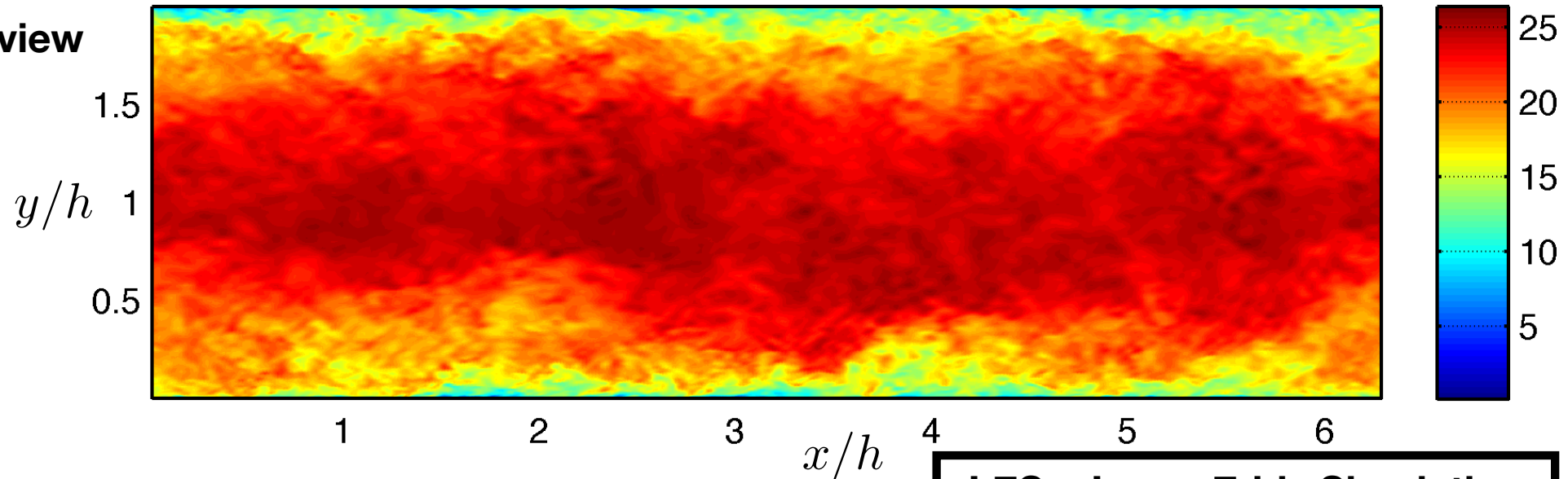
wake function obtained from the DNS



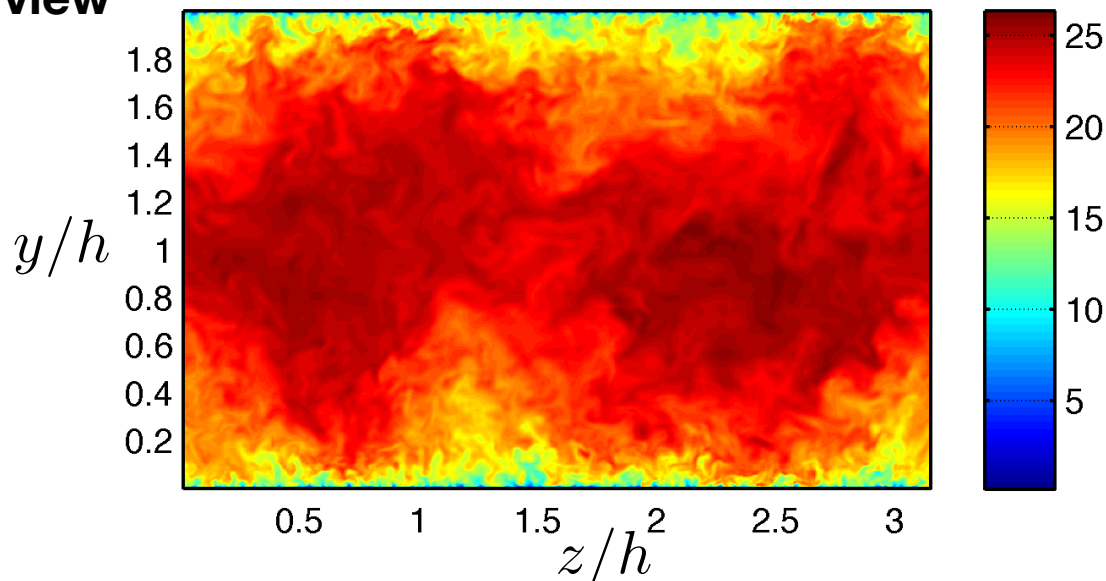
wall-resolved LES of a channel flow

snapshot of instantaneous u field $Re_\tau \stackrel{\text{def}}{=} \frac{h \bar{u}_\tau}{\nu} = 2000$ $Re_d \stackrel{\text{def}}{=} \frac{(2h) \bar{u}_m}{\nu} \simeq 87200$

side view



cross view

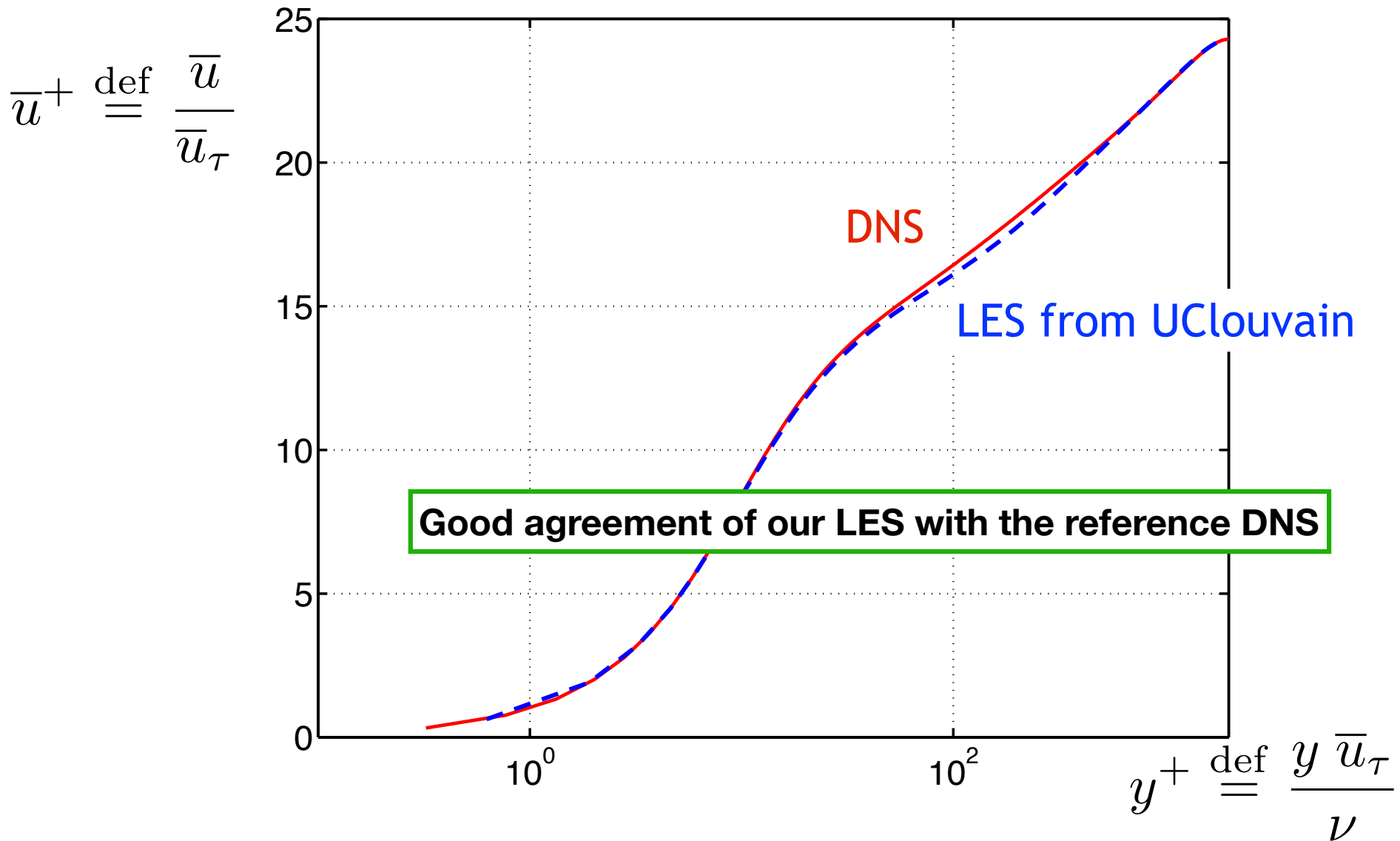


LES = Large Eddy Simulation

**not all the scales are resolved,
yet many scales are !**

**hence the need to model the
effect of the sub-grid scales
(SGS) on the resolved scales**

LES of a channel flow: velocity profile



Thermally developed turbulent channel flow

mean flow velocity : $\bar{u}_m = \frac{1}{h} \int_0^h \bar{u}(y) dy$

**mean flow temperature
(cup mixing temperature) :** $\bar{T}_m = \frac{1}{\bar{u}_m h} \int_0^h \bar{T}(y) \bar{u}(y) dy$

dimensionless temperature profile : $\frac{(\bar{T} - \bar{T}_w)}{(\bar{T}_m - \bar{T}_w)}$

thermally developed flow = this dimensionless profile is solely function of y :

$$\frac{\partial}{\partial x} \left(\frac{(\bar{T} - \bar{T}_w)}{(\bar{T}_m - \bar{T}_w)} \right) = 0$$



$$\left(\frac{\partial \bar{T}}{\partial x} - \frac{d\bar{T}_w}{dx} \right) = \frac{(\bar{T} - \bar{T}_w)}{(\bar{T}_m - \bar{T}_w)} \left(\frac{d\bar{T}_m}{dx} - \frac{d\bar{T}_w}{dx} \right)$$

Thermally developed turbulent channel flow

$$\left(\frac{\partial \bar{T}}{\partial x} - \frac{d\bar{T}_w}{dx} \right) = \frac{(\bar{T} - \bar{T}_w)}{(\bar{T}_m - \bar{T}_w)} \left(\frac{d\bar{T}_m}{dx} - \frac{d\bar{T}_w}{dx} \right)$$

Case with constant heat flux density imposed at the wall :

$$\frac{\partial \bar{T}}{\partial x} = \frac{d\bar{T}_m}{dx} = \frac{d\bar{T}_w}{dx} = \text{Constant}$$

Case with constant temperature imposed at the wall :

$$\frac{\partial \bar{T}}{\partial x} = \frac{(\bar{T} - \bar{T}_w)}{(\bar{T}_m - \bar{T}_w)} \frac{d\bar{T}_m}{dx}$$

to put into the equation for the temperature profile :

$$\bar{u} \frac{\partial \bar{T}}{\partial x} = \frac{1}{c} (\nu + \nu_t) \left(\frac{\partial \bar{u}}{\partial y} \right)^2 + \frac{\partial}{\partial y} \left((\alpha + \alpha_t) \frac{\partial \bar{T}}{\partial y} \right)$$

Fully developed turbulent boundary layer

$\bar{v} \neq 0$ but is much smaller than the external forcing velocity $\bar{u}_e(x)$

$\bar{u}$, $\overline{u'u'}$, $\overline{v'v'}$, $\overline{w'w'}$, $\bar{k}$, $\overline{u'v'}$ also vary in x but slowly

$$\frac{\partial \bar{u}}{\partial x} + \frac{\partial \bar{v}}{\partial y} = 0$$

$$\bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} = -\frac{\partial \bar{P}}{\partial x} - \frac{\partial}{\partial x} (\overline{u'u'}) + \frac{\partial}{\partial y} \left(\nu \frac{\partial \bar{u}}{\partial y} - \overline{u'v'} \right)$$

$$0 = -\frac{\partial \bar{P}}{\partial y} - \frac{\partial}{\partial y} (\overline{v'v'}) \quad \Rightarrow \quad \bar{P} + \overline{v'v'} = \bar{P}_e(x)$$

$$\Rightarrow \quad \frac{\partial \bar{P}}{\partial x} = \frac{d\bar{P}_e}{dx} - \frac{\partial}{\partial x} (\overline{v'v'})$$



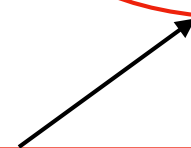
Fully developed turbulent boundary layer

$$\bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} + \frac{\partial}{\partial x} (\overline{u'u'} - \overline{v'v'}) = -\frac{d\bar{P}_e}{dx} + \frac{\partial}{\partial y} \left(\nu \frac{\partial \bar{u}}{\partial y} - \overline{u'v'} \right)$$

Bernoulli equation : $\bar{P}_e(x) + \frac{\bar{u}_e^2(x)}{2} = \text{Constant}$


$$-\frac{d\bar{P}_e}{dx} = \bar{u}_e \frac{d\bar{u}_e}{dx}$$


$$\bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} + \frac{\partial}{\partial x} (\overline{u'u'} - \overline{v'v'}) = \bar{u}_e \frac{d\bar{u}_e}{dx} + \frac{\partial}{\partial y} \left(\nu \frac{\partial \bar{u}}{\partial y} - \overline{u'v'} \right)$$



this is the « Townsend term »

Fully developed turbulent boundary layer

The boundary layer equations for the velocity field are finally taken as :

$$\frac{\partial \bar{u}}{\partial x} + \frac{\partial \bar{v}}{\partial y} = 0$$

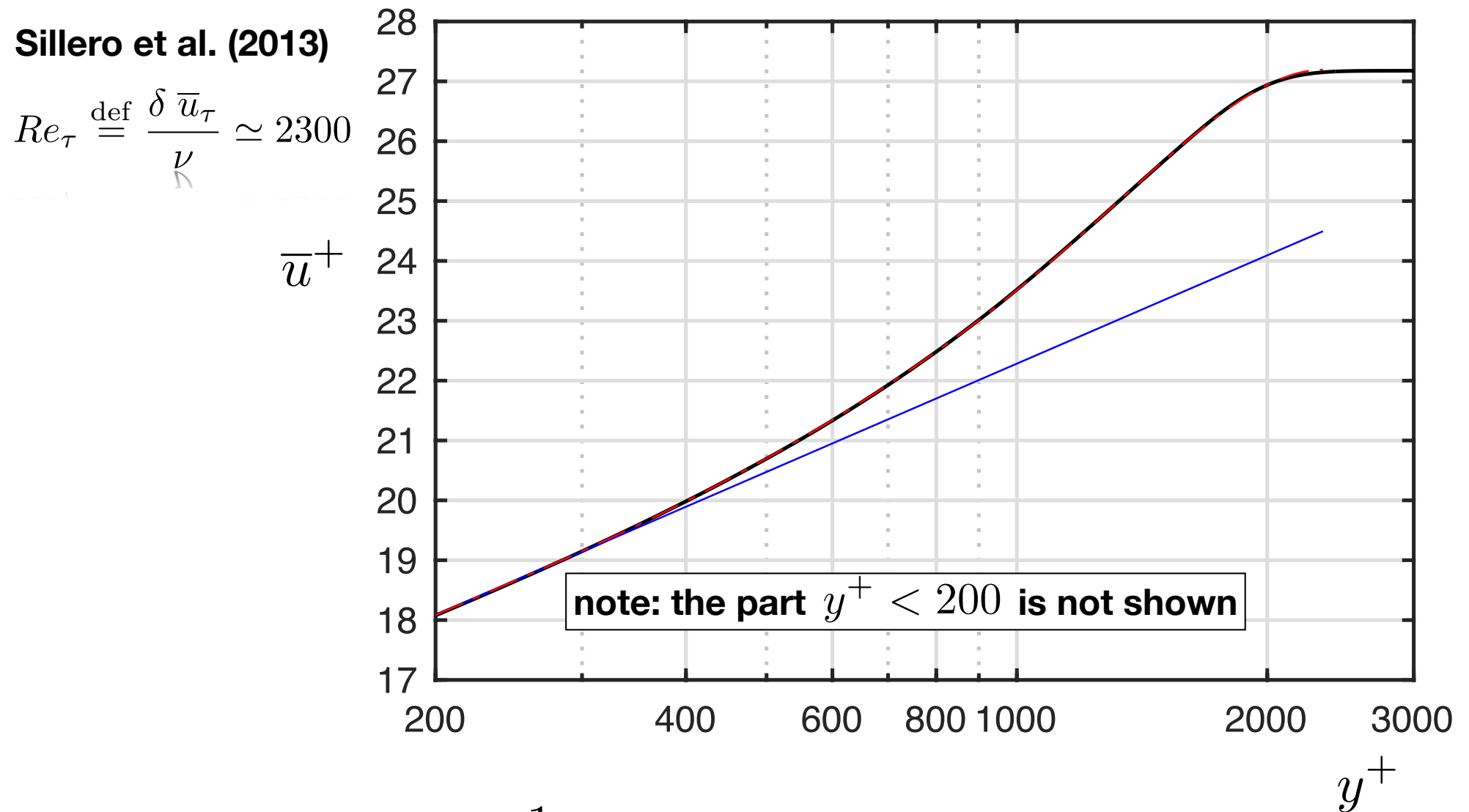
$$\bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} = \bar{u}_e \frac{d\bar{u}_e}{dx} + \frac{\partial}{\partial y} \left((\nu + \nu_t) \frac{\partial \bar{u}}{\partial y} \right)$$

(thus neglecting the variation in x of the Townsend term !)

As to the boundary layer equation for the temperature field, it is obtained as :

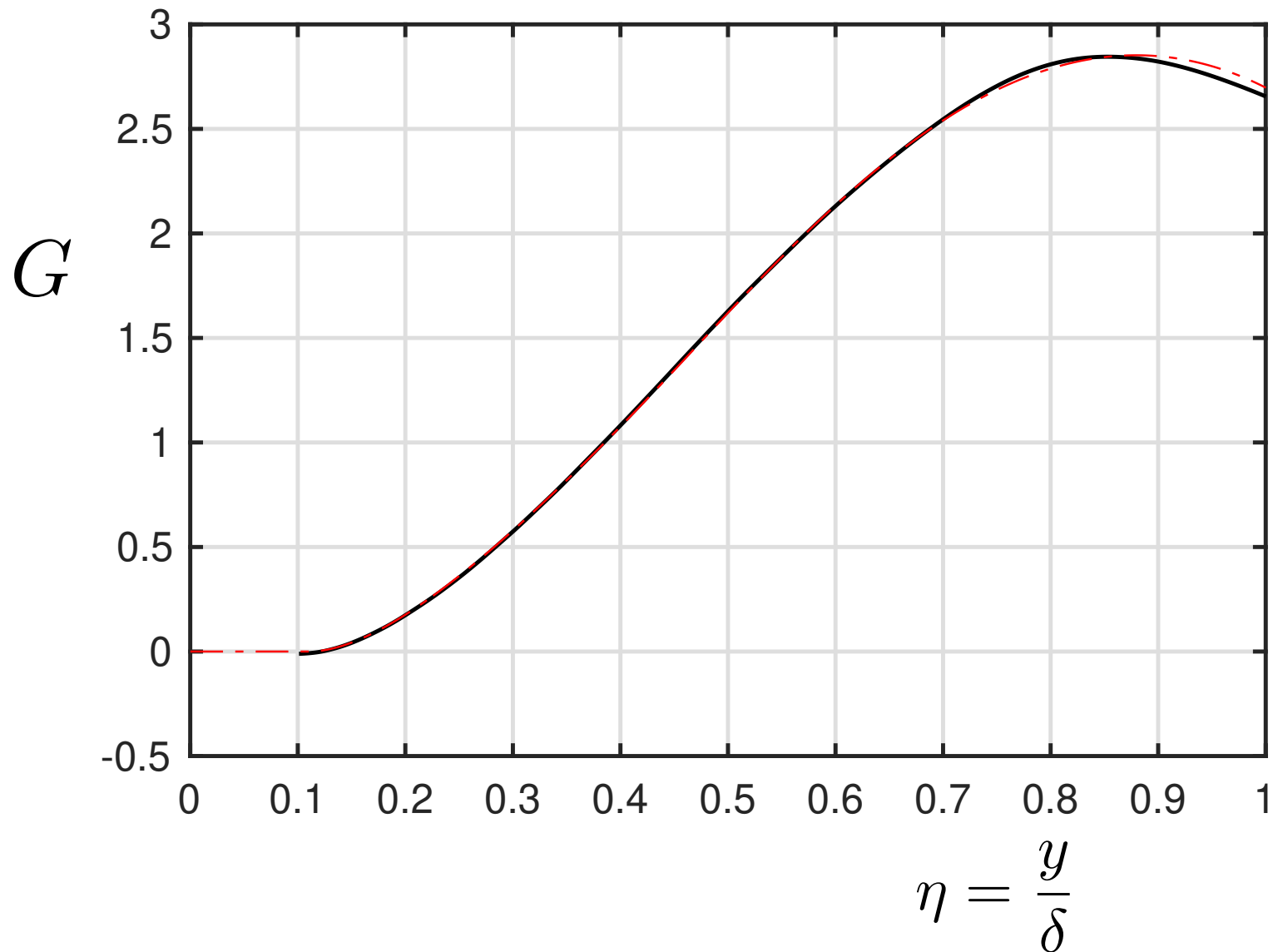
$$\bar{u} \frac{\partial \bar{T}}{\partial x} + \bar{v} \frac{\partial \bar{T}}{\partial y} = \frac{1}{c} (\nu + \nu_t) \left(\frac{\partial \bar{u}}{\partial y} \right)^2 + \frac{\partial}{\partial y} \left((\alpha + \alpha_t) \frac{\partial \bar{T}}{\partial y} \right)$$

velocity profile obtained from a DNS

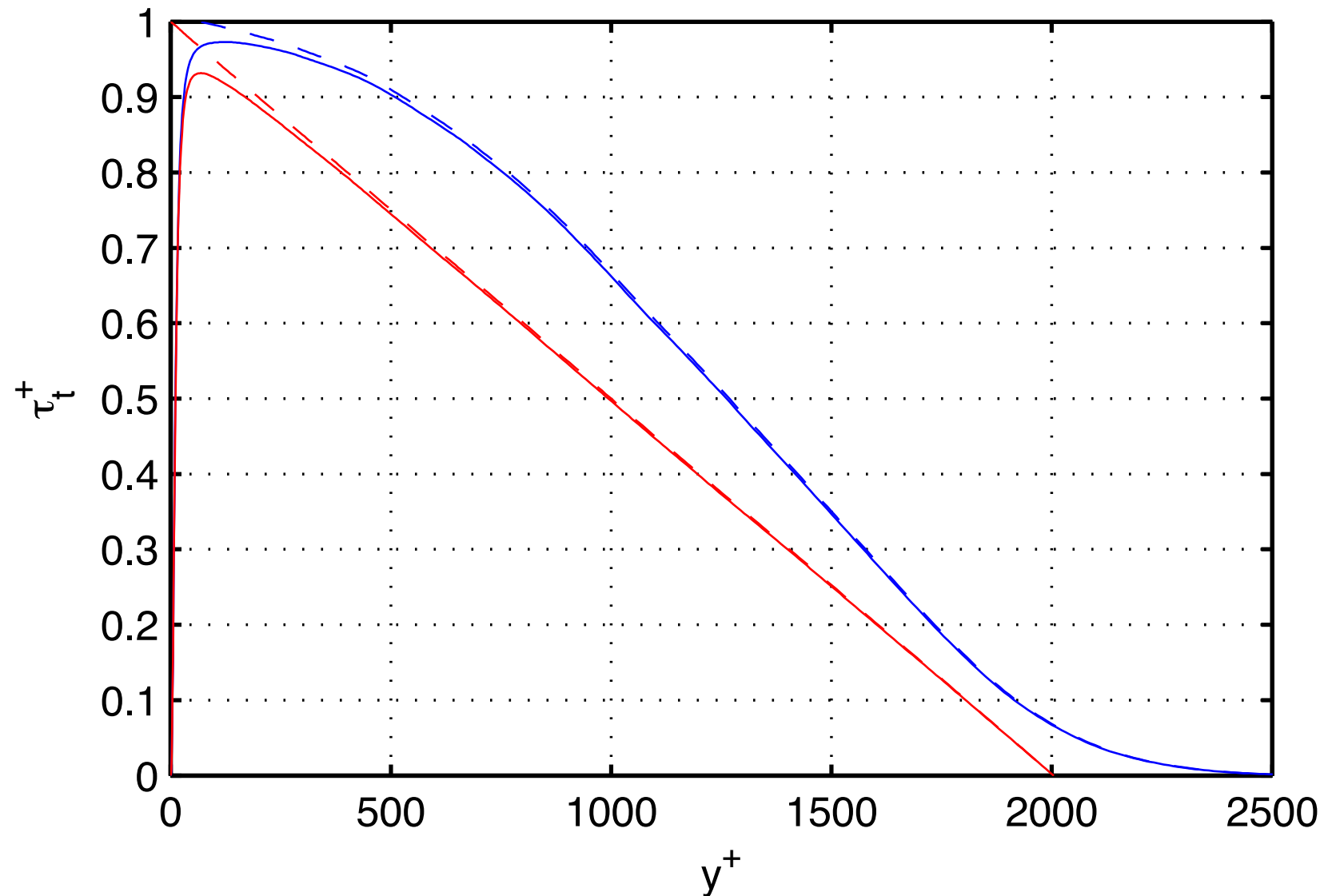


log law (in blue) : $\bar{u}^+ = \frac{1}{\kappa} \log y^+ + C$

wake function obtained from a DNS



Turbulent shear stress profile: channel versus boundary layer



Adding gravity

- **Case with constant density:**

- it suffices to include the hydrostatic pressure into the definition of the kinematic pressure:

$$P \stackrel{\text{def}}{=} \frac{(p + \rho g y)}{\rho} = \frac{p}{\rho} + g y$$

- then nothing changes: we obtain the same Navier-Stokes equations for incompressible flows as before, and thus also the same RANS equations, and the same physics.

Adding gravity

- **Case with small variations of density** (yet still assuming that the flow remains incompressible = Boussinesq approximation):

$$\rho = \rho_0 (1 - \beta (T - T_0))$$

- we then define the kinematic pressure as:

$$P \stackrel{\text{def}}{=} \frac{(p + \rho_0 g y)}{\rho_0} = \frac{p}{\rho_0} + g y$$

- we obtain an added « buoyancy term » on the RHS of the momentum equation for v : $\beta (T - T_0) g$

- and thus also in the RANS equation for $\bar{v}$: $\beta (\bar{T} - T_0) g$