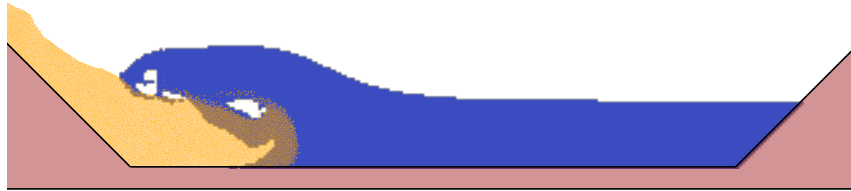


Numerical methods for granular media in multiphase fluids



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CHAPTER



2

Non-Smooth Dynamics in Granular Physics

If you can't explain it simply, you don't understand it well enough.

— Albert Einstein.

Granular materials consist of macroscopic particles that interact through contact forces, friction, and collisions. Unlike fluids or conventional solids, granular matter lacks a clear scale separation between microscopic interactions and macroscopic behaviour, making it difficult to describe with classical continuum mechanics [65, 66, 67]. In particular, defining consistent constitutive laws, that relate stress and strain in a predictive way, is challenging due to the discrete nature of the medium, history dependence, and strong heterogeneities in force transmission [68]. These limitations motivate the use of numerical approaches that resolve grain-scale interactions directly.

One such approach is the Discrete Element Method (DEM) originally developed by Cundall and Strack [36] for geomechanical problems. Particles are modeled as rigid bodies whose motion and contact interactions are governed by Newton-Euler's laws of motion. By treating grains as rigid bodies, DEM avoids the need to define bulk constitutive relationships, instead computing macroscopic responses as an emergent result of contact-level physics. Each grain's

position and orientation evolve according to forces arising from inter-particle contacts, gravity, and possibly fluid interactions. This particle-scale resolution provides a natural way to capture phenomena such as jamming, segregation, dilatancy, and energy dissipation. These features are difficult to encode in continuum models [69, 70, 71].

Traditional implementations of the Discrete Element Method (DEM) often rely on smooth contact laws, whereby small overlaps between particles are used to model elastic or viscoelastic behaviour [72]. This approach, known as the soft-sphere model, enables efficient explicit time integration, which is particularly well-suited for GPU-based parallelization [37]. While effective for loosely packed or weakly interacting granular systems, soft-sphere DEM can encounter stability and efficiency issues when applied to densely packed, stiff, or quasi-rigid materials. In these regimes, large numbers of simultaneous contacts and rapidly changing contact networks can lead to numerical instabilities and require prohibitively small time steps to maintain accuracy and robustness. To overcome these limitations, the Non-Smooth Contact Dynamics (NSCD) approach has been developed [73, 74, 75]. NSCD treats particles as perfectly rigid bodies and resolves contact interactions through complementarity conditions or variational inequalities, which enforce non-penetration and Coulomb friction constraints directly, without relying on artificial dissipation or elastic deformation. This framework is particularly advantageous for modelling dense granular flows, where persistent contacts, sudden transitions between sticking and sliding, and highly intermittent force chains are common. Unlike soft-sphere, NSCD is capable of resolving large contact networks and discontinuous motion in a stable and consistent manner, without requiring small time steps or the regularization of contact laws. This makes it especially suitable for applications such as quasi-static deformation, compaction, or impact-driven granular dynamics, where the accurate treatment of contact conditions is critical.

In real-world scenarios, granular flows often occur in the presence of an interstitial fluid, such as air or water. The interaction between the grains and the fluid can dramatically influence the flow behaviour due to added mass effects, buoyancy, drag, lubrication, and variations in pore pressure. The force applied by the fluid to each grain can be incorporated directly into the DEM framework through an external fluid-grain interaction force. This external force can be explicitly integrated into the NSCD framework, enabling the simulation of immersed granular flows. Additionally, since overlapping particles are not allowed, the numerical scheme naturally satisfies particle volume conservation.

The Discrete Element Method (DEM) can be summarised as follows. Algorithm 1 provides an overview of the key steps involved in a single time step of the DEM. First, the *free velocity* of each particle is computed by integrating the external forces (such as gravity and fluid-grain interactions) over the time step, assuming no contact interactions. Next, the *contact detection* step identifies all potential contacts between particles and between particles and

boundaries. These contacts are used to formulate the contact constraints, such as non-penetration and frictional sliding. Once contacts have been detected, the *contact problem* is solved using a Non-Smooth Contact Dynamics (NSCD) approach. This involves computing the *contact impulse* necessary to enforce the contact constraints. This impulse is then applied to adjust the velocities so that they satisfy both Newton’s laws and the contact conditions. Finally, the particle positions and orientations are updated using the post-contact velocities, thus completing the time step. These updated states are then used to advance to the next time step in the simulation.

In this chapter, we present the fundamentals of the Discrete Element Method as applied to the particulate phase in fluid–granular systems, with a focus on its non-smooth dynamics formulation. We detail the theoretical framework of NSCD and its numerical implementation.

Algorithm 1: Iteration of a DEM Solver.

Require: $\mathbf{q}^n, \mathbf{v}^n, \mathbf{f}_e^n, \Delta t$

$\mathbf{v}^- \leftarrow \mathbf{v}^n + \Delta t \mathbf{f}_e^n$

$\mathcal{C} \leftarrow$ Contact Detection

$\mathbf{p}^{n+1}, \mathbf{v}^{n+1} \leftarrow$ Solve the contacts problem

$\mathbf{q}^{n+1} \leftarrow$ Update the bodies position

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2.1 Non-Smooth Newton–Euler Dynamics

Originally developed for geomechanical applications [36], the Discrete Element Method (DEM) has evolved into a widely adopted numerical approach for simulating the behaviour of granular materials. In this framework, each particle is modelled as a rigid body, and its motion is governed by the Newton–Euler equations. The discontinuous nature of collisions and contact interactions necessitates a non-smooth formulation of the dynamics. Generalised configuration variables are introduced to describe the particle’s position and orientation, providing a comprehensive representation of its motion. This formulation leads to a concise set of equations that govern the particle’s dynamics, including both linear and angular momentum balances which will be suitable to describe the contact dynamics.

2.1.1 Rigid body kinematics

A rigid body can be described using generalised coordinates, which represent its position $\mathbf{y} \in \mathbb{R}^3$ and its orientation, represented by a unit quaternion $\boldsymbol{\theta} \in \mathbb{H}^4$ in the Euclidean space \mathbb{R}^3 . For two-dimensional systems, a single angle is sufficient to describe the orientation. For numerical purposes, unit quaternions are often preferred to Euler angles or a rotation matrix due to their computational efficiency and numerical stability. The generalised velocity, denoted \mathbf{v} , relates the generalised coordinates to the linear and angular velocities in the body-fixed reference frame:

$$\mathbf{q} = \begin{bmatrix} \mathbf{y} \\ \boldsymbol{\theta} \end{bmatrix}, \quad \mathbf{v} = \begin{bmatrix} \dot{\mathbf{y}} \\ \boldsymbol{\omega} \end{bmatrix} \quad (2.1)$$

where $\dot{\mathbf{y}} \in \mathcal{R}^3$ is the linear velocity and $\boldsymbol{\omega} \in \mathcal{R}^3$ is the angular velocity.

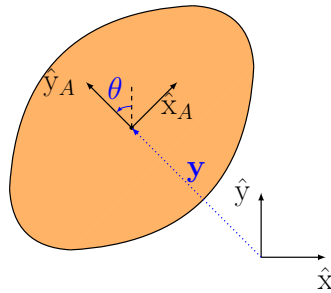


Figure 2.1: Configuration variables of a rigid body.

The time derivative of the orientation, $\boldsymbol{\theta} \in \mathbb{H}^4$ is related to the angular velocity, $\boldsymbol{\omega} \in \mathbb{R}^3$ by:

$$\dot{\boldsymbol{\theta}} = \frac{1}{2} \mathbf{Q}(\boldsymbol{\theta}) \boldsymbol{\omega} \quad (2.2)$$

$$\dot{\boldsymbol{\theta}} = \frac{1}{2} \begin{bmatrix} -\omega_x \theta_x - \omega_y \theta_y - \omega_z \theta_z \\ \omega_x \theta_w + \omega_y \theta_z - \omega_z \theta_y \\ \omega_y \theta_w - \omega_x \theta_z + \omega_z \theta_x \\ \omega_z \theta_w + \omega_x \theta_y - \omega_y \theta_x \end{bmatrix} \quad (2.3)$$

It follows that the time derivative of the generalised coordinates \mathbf{q} can be expressed as:

$$\dot{\mathbf{q}} = \mathbf{T}(\mathbf{q}) \mathbf{v} \quad (2.4)$$

where $\mathbf{T}(\mathbf{q})$ is the transformation matrix from the generalised velocities to the generalised coordinates based on Equation (2.3). The generalised coordinates are defined by six degrees of freedom in three-dimensional space and three degrees of freedom in two-dimensional space. Figure 2.1 illustrates the rigid body configuration.

2.1.2 Non-Smooth Dynamics

The dynamics of a rigid body are governed by the Newton–Euler equations, which describe the motion of the particle in terms of its linear and angular momentum. External impulses act on the particle, which can be due to contact forces, fluid forces, or external forces such as gravity. Figure 2.2 illustrates the motion of a particle subjected to gravity and an external force. To account for the non-smooth nature of contact interactions, the dynamics equations are formulated in a differential form [76, 74]:

$$\mathbf{M} d\mathbf{v} = \mathbf{f}_e dt + \mathbf{f}_q dt + d\mathbf{p} \quad (2.5)$$

where $d\mathbf{v}$ is the differential measure of the velocity, \mathbf{f}_e is the external generalised force, $d\mathbf{p}$ is the generalised impulse and \mathbf{M} is the generalised mass matrix defined as:

$$\mathbf{M} = \begin{bmatrix} m\mathbf{I} & \mathbf{0} \\ \mathbf{0} & \mathbf{J} \end{bmatrix} \quad (2.6)$$

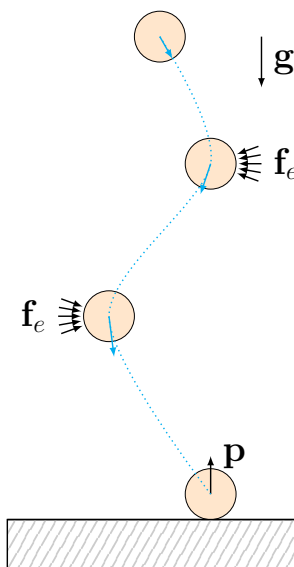


Figure 2.2: A disc subjected to external forces.

where m is the mass of the body, $\mathbf{I} \in \mathbb{R}^{3 \times 3}$ is the identity matrix, $\mathbf{J} \in \mathbb{R}^{3 \times 3}$ is the inertia matrix and $\mathbf{0} \in \mathbb{R}^{3 \times 3}$ is the null matrix. The generalised external force \mathbf{f}_e includes the external forces and moments acting on the particle, such as gravity or fluid forces. Contact forces and moments are included in the generalised impulse. Gyroscopic moments are included in the generalised quadratic force \mathbf{f}_q , which is defined as:

$$\mathbf{f}_q = \begin{bmatrix} \mathbf{0} \\ -\boldsymbol{\omega} \times \mathbf{J}\boldsymbol{\omega} \end{bmatrix} \quad (2.7)$$

For a sphere or bidimensional system, the quadratic force is zero because the inertia matrix is diagonal. This term will be omitted from the following equations, since only two-dimensional systems and symmetric particles will be considered. To model contact interactions, inequalities are used to enforce non-penetration and frictional constraints. The Discrete Element Method (DEM) solves the equations of motion for each particle in the system, Equation (2.5), neighboring particles interact through contact forces and moments.

In order to address the discontinuous nature of the contact impulse, the generalised velocity \mathbf{v} is assumed to be a right continuous locally bounded variation function. This means that the measure of $]t_0, t]$ by $d\mathbf{v}$ is

$$\int_{]t_0, t]} d\mathbf{v} = \mathbf{v}(t) - \mathbf{v}(t_0). \quad (2.8)$$

The kinematics and dynamics of a rigid body in the non-smooth framework can be expressed as:

$$\mathbf{q}(t) - \mathbf{q}(t_0) = \int_{]t_0, t]} \mathbf{T}(\mathbf{q}) d\mathbf{v}, \quad (2.9)$$

$$\mathbf{M}(\mathbf{v}(t) - \mathbf{v}(t_0)) = \int_{]t_0, t]} \mathbf{f}_e dt + \mathbf{p}. \quad (2.10)$$

The unknowns in these equations are the generalised coordinates \mathbf{q} , the generalised velocities \mathbf{v} , and the generalised impulses \mathbf{p} . In practice, the external force \mathbf{f}_e is integrated explicitly, and the *free velocity* \mathbf{v}^- is defined as the velocity the body would get in the absence of contact interactions:

$$\mathbf{v}^- = \mathbf{v}(t_0) + \mathbf{M}^{-1}\mathbf{f}_e\Delta t, \quad (2.11)$$

where \mathbf{M} is the mass matrix and Δt is the time step. It leads to the following expression for the body dynamics:

$$\mathbf{M}(\mathbf{v}^+ - \mathbf{v}^-) = \mathbf{p}, \quad (2.12)$$

where \mathbf{v}^+ and \mathbf{v}^- denote the post and pre-impact velocities, respectively. The unknown quantities in this equation are the post-impact velocity \mathbf{v}^+ and the contact impulse \mathbf{p} . The next section introduces the *Contact Dynamics* method, which uses non-overlapping and frictional contact constraints to determine these quantities.

2.2 Contact Dynamics

The core principle of the Non-Smooth Contact Dynamics (NSCD) method is to treat contact events as discontinuous impulse. All contacts occurring within a time step are considered instantaneous and are solved simultaneously. This approach enables the simulation of contact dynamics even when the duration of contact tends toward zero. However, a fundamental trade-off of this method is the loss of uniqueness of the solution, meaning that multiple valid outcomes may exist for a given configuration. The method can be divided into three main stages:

Contact Detection involves identifying potential contact pairs. This is done by computing the distance between particles and checking whether they are sufficiently close to interact. A naive approach would involve checking all possible particle pairs, but this becomes computationally infeasible for large systems. To address this issue, tree-based spatial partitioning algorithms are employed to efficiently detect potential contacts using particle bounding boxes. As a consequence of the detection step, a time step restriction arises from this stage to ensure accurate detection.

Local Contact Resolution Once potential contacts have been identified, each one is treated individually. A local contact frame is then defined at each contact point to serve as the reference basis for expressing contact conditions. Within this frame, non-overlapping constraints, friction, and inelastic collision conditions are formulated as inequalities. The contact dynamics are therefore expressed in terms of complementarity conditions within this local frame.

Global Resolution Solving the full contact problem involving multiple interacting bodies requires iterative algorithms. Since multiple contacts can act simultaneously on a single body, the contact problem is inherently coupled and nonlinear. To handle these interdependencies and ensure a consistent global solution, iterative schemes are employed to progressively converge towards a stable configuration. Commonly used methods include the *Nonlinear Gauss–Seidel* algorithm [40, 77], *Conjugate Gradient*

methods [78], and optimisation-based approaches such as the *Accelerated Gradient Scheme* [39].

This section details the procedures for both local and global contact resolution. First, we introduce the kinematics of contact, including the definition of the contact frame and the associated transformation matrix. Next, we present Signorini's condition, which enforces the non-penetration constraint between interacting bodies. This is followed by a discussion of Coulomb's friction law, which governs the tangential forces that arise at contact points. Finally, we describe the Gauss–Seidel algorithm, an efficient iterative method for solving the global multi-contact problem.

2.2.1 Contact Kinematics

To express the equations of motion in the local frame, we introduce a local frame associated with the contact point. Let \mathbf{n} be the unit normal vector to the contact plane and \mathbf{s} and \mathbf{t} be the two tangential vectors in the contact plane. The normal vector is defined as the unit vector pointing from the two closest point between body A and body B . It leads that the gap between the two bodies is given by,

$$g(t) = \mathbf{n} \cdot (\mathbf{y}_A^c - \mathbf{y}_B^c), \quad (2.13)$$

where \mathbf{y}_A^c is the closest point from body A to body B and \mathbf{y}_B^c is the closest point to body A on body B . Figure 2.3 illustrates the local frame associated with the contact point.

We seek to express the equations of motion in the local frame associated with the contact point. The absolute velocity at the contact point in the global frame is given by :

$$\mathbf{v} = \dot{\mathbf{y}} + \boldsymbol{\omega} \times \mathbf{y}^c, \quad (2.14)$$

$$= [\mathbf{I} \quad \mathbf{R}] \begin{bmatrix} \dot{\mathbf{y}} \\ \boldsymbol{\omega} \end{bmatrix} \quad (2.15)$$

$$(2.16)$$

where \mathbf{I} is the identity matrix, \mathbf{R} the skew-symmetric matrix use to express the vector cross-product as a matrix-vector product:

$$\mathbf{R} = \begin{bmatrix} 0 & z^c & -y^c \\ -z^c & 0 & x^c \\ y^c & -x^c & 0 \end{bmatrix}, \quad (2.17)$$

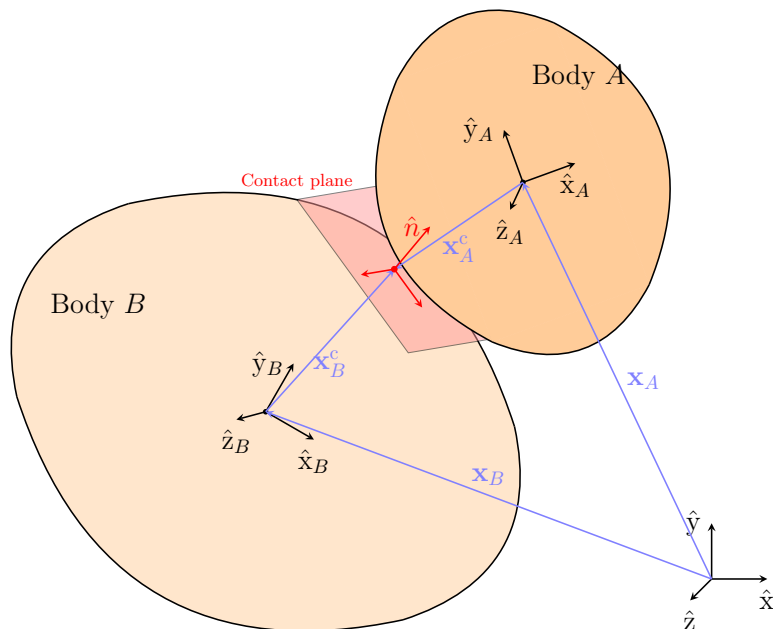


Figure 2.3: Local frames associated with the contact problem.

The absolute velocity and the contact impulse, respectively denoted by \mathbf{u} and \mathbf{r} , can be expressed in the local frame as:

$$\mathbf{u} = \mathbf{H}^T \mathbf{v}, \quad (2.18)$$

$$\mathbf{p} = \mathbf{H} \mathbf{r}, \quad (2.19)$$

where \mathbf{H} maps the global frame to the local frame at the contact point, defined as:

$$\mathbf{H} = \begin{bmatrix} \mathbf{I} \\ \mathbf{R} \end{bmatrix} \mathbf{B}, \quad \mathbf{B} = [\mathbf{n} \quad \mathbf{s} \quad \mathbf{t}], \quad (2.20)$$

where \mathbf{B} is change of basis matrix from the local frame to the global frame. It should be recall that the contact point and The transformation matrix \mathbf{H} are not known a priori as they depends on the bodies configuration.

Over a singleton $\{t^c\}$, the jump velocity at the contact point, Equation

(2.12), can be expressed in the local frame as:

$$\mathbf{M}(\mathbf{v}^+ - \mathbf{v}^-) = \mathbf{p}, \quad (2.21)$$

$$\mathbf{v}^+ - \mathbf{v}^- = \mathbf{M}^{-1}\mathbf{p}, \quad (2.22)$$

$$\mathbf{H}^T(\mathbf{v}^+ - \mathbf{v}^-) = \mathbf{H}^T\mathbf{M}^{-1}\mathbf{H}\mathbf{r}, \quad (2.23)$$

$$\mathbf{u}^+ - \mathbf{u}^- = \mathbf{H}^T\mathbf{M}^{-1}\mathbf{H}\mathbf{r}, \quad (2.24)$$

It is convenient to introduce the Delassus operator \mathbf{W} which describes the change of velocity at the contact point due to the contact impulse,

$$\mathbf{u}^+ - \mathbf{u}^- = \mathbf{W}\mathbf{r}, \quad \mathbf{W} = \mathbf{H}^T\mathbf{M}^{-1}\mathbf{H}, \quad (2.25)$$

At the contact point between two bodies A and B , the dynamics at the contact point is expressed as:

$$\mathbf{u}_A^+ = \mathbf{u}_A^- + \mathbf{W}_A\mathbf{r}, \quad \mathbf{W}_A = \mathbf{H}_A^T\mathbf{M}_A^{-1}\mathbf{H}_A \quad (2.26)$$

$$\mathbf{u}_B^+ = \mathbf{u}_B^- - \mathbf{W}_B\mathbf{r}, \quad \mathbf{W}_B = \mathbf{H}_B^T\mathbf{M}_B^{-1}\mathbf{H}_B \quad (2.27)$$

Let $\delta\mathbf{u} = \mathbf{u}_A - \mathbf{u}_B$ be the relative velocity at the contact point. The dynamics at the contact point is:

$$\delta\mathbf{u}^+ = \delta\mathbf{u}^- + \mathbf{W}\mathbf{r}, \quad \mathbf{W} = \mathbf{W}_A + \mathbf{W}_B \quad (2.28)$$

Regarding this equation, both the contact impulse \mathbf{r} and the relative velocity $\delta\mathbf{u}^+$ are unknown. In three-dimensional space, six unknowns have to be solved, the contact dynamics, Equation (2.28) only provides three equations. To close the system, non-overlapping conditions are imposed at the contact point and constitutive laws are introduced to model tangential reactions with inelastic collisions.

2.2.2 Signorini's Condition

Non-smooth methods seek to prevent overlapping between two bodies. This is why they are also referred as hard-sphere methods. To ensure non-overlapping, the gap, Equation (2.13), is set to be non-negative

$$g(t) \geq 0, \quad (2.29)$$

This condition is referred as the unilateral constraint. As two bodies are close to one another without contact, the gap is positive and the normal contact impulse is zero. When the two bodies are in contact, the gap is zero and the

normal contact impulse is positive. This situation is referred as the Signorini's condition:

$$g(t) \geq 0, \quad (\text{Impenetrability}), \quad (2.30)$$

$$r_n(t) \geq 0, \quad (\text{No attraction}), \quad (2.31)$$

$$r_n(t)g(t) = 0, \quad (\text{Signorini's condition}), \quad (2.32)$$

where r_n is the normal contact impulse. Figure 2.4a illustrates the Signorini's condition.

2.2.3 Frictional Contact Law

In order to fully describe the contact dynamics, constitutive laws are required to model the contact forces. Inelastic shock and Coulomb's friction law are the ones considered in this work. The inelastic shock states that if a contact occurs, the contact normal velocity vanishes after the contact,

$$g(t) = 0 \Rightarrow r_n(t) \geq 0, \quad \delta u_n^+ = 0. \quad (2.33)$$

Coulomb's friction law states that the tangential contact impulse is opposite to the tangential velocity at the contact point and that its magnitude belongs to the friction cone with a friction coefficient μ . Figure 2.4b illustrates the Coulomb's friction law for a two-dimensional system. In the contact frame, the Coulomb's friction law can be expressed as:

$$\|\mathbf{r}_t\| \leq \mu r_n, \quad (2.34)$$

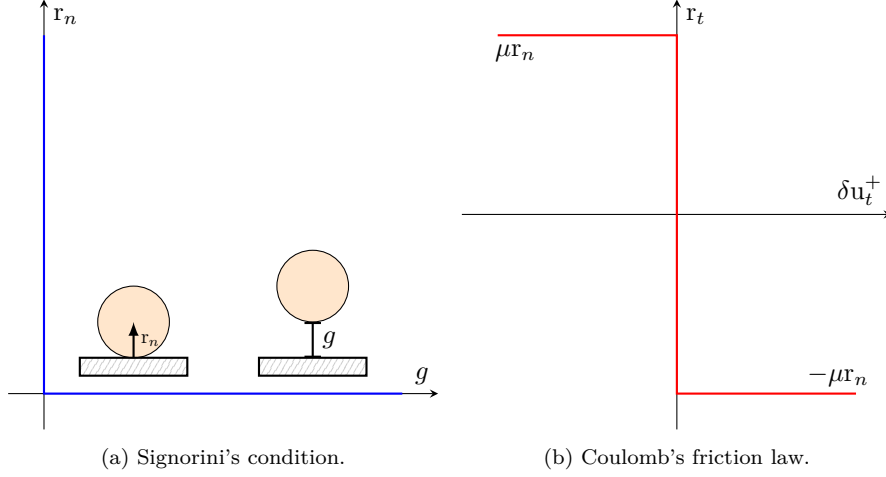
$$\|\delta \mathbf{u}_t^+\| \neq 0 \Rightarrow \mathbf{r}_t = -\mu r_n \frac{\delta \mathbf{u}_t^+}{\|\delta \mathbf{u}_t^+\|}. \quad (2.35)$$

where \mathbf{r}_t is the tangential contact impulse and $\delta \mathbf{u}_t^+$ is the tangential velocity at the contact point defined as:

$$\mathbf{r}_t = \mathbf{r} - r_n \mathbf{n}, \quad \delta \mathbf{u}_t^+ = \delta \mathbf{u}^+ - \delta u_n^+ \mathbf{n}. \quad (2.36)$$

2.2.4 A Brief Summary

For a given contact point, the contact impulse and the relative velocity are determined to satisfy the contact dynamics, the Signorini condition (impenetrability), Coulomb's friction law (Coulomb's cone and slipping), and the inelastic impact condition (no restitution).



$$\begin{aligned}
 \delta \mathbf{u}^+ &= \delta \mathbf{u}^- + \mathbf{W} \mathbf{r}, & (\text{Contact Dynamics}) \\
 g(t) &\geq 0, & (\text{Impenetrability}), \\
 r_n(t) &\geq 0, & (\text{No attraction}), \\
 r_n(t)g(t) &= 0, & (\text{Signorini's condition}), \\
 g(t) = 0 &\Rightarrow r_n(t) \geq 0, \delta u_n^+ = 0, & (\text{No restitution}), \\
 \|\mathbf{r}_t\| &\leq \mu r_n, & (\text{Coulomb's cone}) \\
 \|\delta \mathbf{u}_t^+\| \neq 0 &\Rightarrow \mathbf{r}_t = -\mu r_n \frac{\delta \mathbf{u}_t^+}{\|\delta \mathbf{u}_t^+\|}, & (\text{Slipping}).
 \end{aligned} \tag{2.37}$$

The complete contact problem is inherently nonlinear, so finding a solution is not trivial. Fortunately, in two-dimensional problems, an explicit solution can often be obtained using a graphical interpretation of the contact laws. This allows for a direct construction of the contact impulse that satisfies the Signorini condition, Coulomb's friction law, and the inelastic impact condition. However, in three-dimensional configurations, the contact space becomes more complex, and an iterative procedure is typically required to find the correct contact impulse at each contact point. The following paragraph illustrates how the contact solution can be obtained explicitly in a bidimensional problem.

A bidimensional example — Let us assume an active contact between two bidimensional bodies, *i.e.* the gap is zero. The contact solution is the relative velocity and the contact impulse such that both the contact dynamics and the Coulomb's friction law are satisfied. The solution can be found graphically by determining the intersection between the contact dynamics and the

Coulomb friction law, as illustrated in Figure 2.5. To discuss this intersection, we introduce the stick impulse, which is the impulse such that the relative velocity vanishes:

$$\mathbf{r}^{\text{stick}} = -\mathbf{W}^{-1}\delta\mathbf{u}^-. \quad (2.38)$$

If the stick solution is inside the friction cone, then the contact is in stick mode and the contact impulse is equal to the stick impulse. If the stick solution is outside the friction cone, then the contact is in slip mode and the contact impulse is equal to the boundary of the friction cone. The sign of the stick tangential impulse determines the direction of the slip either forward or backward,

$$\text{if } r_t^{\text{stick}} - \mu r_n^{\text{stick}} < 0, \quad \text{then } r_t = -\mu r_n \quad (2.39)$$

$$\text{if } r_t^{\text{stick}} + \mu r_n^{\text{stick}} > 0, \quad \text{then } r_t = \mu r_n \quad (2.40)$$

Finally, the tangential contact impulse is injected into the contact dynamics, a 2×2 system of linear equations is obtained,

$$\begin{bmatrix} 0 \\ \delta\mathbf{u}_t^+ \end{bmatrix} = \begin{bmatrix} \mathbf{W}_{nn} & \mathbf{W}_{nt} \\ \mathbf{W}_{tn} & \mathbf{W}_{tt} \end{bmatrix} \begin{bmatrix} r_n \\ r_t \end{bmatrix} + \begin{bmatrix} \delta\mathbf{u}_n^- \\ \delta\mathbf{u}_t^- \end{bmatrix}. \quad (2.41)$$

From this system, the tangential velocity and the normal impulse can be isolated and thus, the contact problem has been fully solved.

In this section, we present the Non-Smooth Contact Dynamics (NSCD) framework for modelling a single collision between two bodies. The dynamics are described in a local frame associated with the contact point. To ensure non-penetration at all times, Signorini's condition is applied. To close the system of equations, constitutive laws are introduced to model the contact forces—specifically, the inelastic impact condition and Coulomb's friction law. However, in practical scenarios, multiple contacts typically occur within a single time step. Resolving these interactions requires solving a global multi-contact problem. The following section details the iterative algorithm used to compute a consistent set of contact impulses that satisfies all constraints.

2.3 When One Contact Isn't Enough

In most realistic scenarios, multiple contacts occur simultaneously within a single time step. The objective is to determine a global contact impulse such that each local contact problem satisfies the conditions outlined in Equation (2.2.4).

Given that the current implementation is sequential, we assume contacts are resolved one after another. However, this resolution introduces a challenge: solving one contact updates the velocities of the involved bodies, which

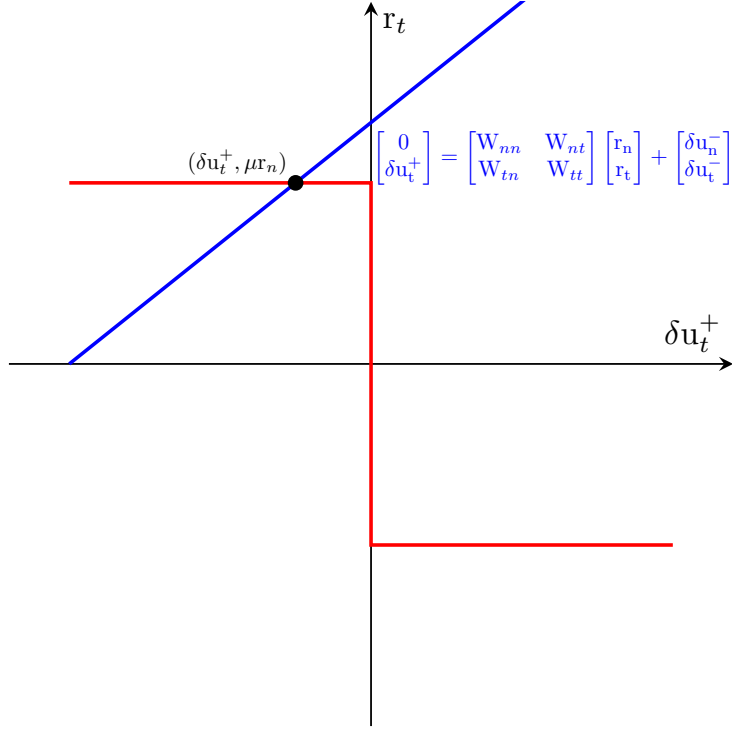


Figure 2.5: Coulomb's friction contact solution procedure.

may, in turn, affect other previously solved contacts. To address this coupling, iterative schemes are employed to achieve convergence. Common methods include the *Nonlinear Gauss–Seidel* algorithm [40, 77] and *Conjugate Gradient methods* [78]. Optimisation-based approaches such as the *Accelerated Gradient Scheme* [39] have also shown promise, offering efficient convergence properties.

In this work, we adopt the Nonlinear Gauss–Seidel method to solve the multi-contact problem. Algorithm 2 summarizes the procedure. Let us assume that all detected contacts are stored in a list \mathcal{C} . The algorithm begins with an initial guess for the contact impulses, denoted \mathbf{r}_0 . It then iteratively loops through each contact in \mathcal{C} . For each contact, the previously applied contact impulse is removed from the bodies. Then the local contact problem is solved using Equation (2.2.4), yielding an updated contact impulse. This contact impulse is then re-applied to the bodies involved in the contact. To assess local convergence, the displacement at the contact point is compared to that from the previous iteration. If the change is below a specified tolerance ξ , the contact is considered to have converged and the previous contact impulse is applied. The algorithm terminates once all contacts satisfy the local convergence criterion.

Algorithm 2: Non-Linear Gauss Seidel Loop.

```

Data: Initial guess  $\mathbf{r}_0$ 
 $k = 0$ ;
converged = False;
while not converged do
  for Each contact  $\alpha \in \mathcal{C}$  do
     $k += 1$ ;
     $\mathbf{v}_0^\alpha, \mathbf{v}_1^\alpha \rightarrow$  Unapply the contact impulse  $\mathbf{r}_{k-1}^\alpha$ ;
     $\delta \mathbf{u}_k^\alpha, \mathbf{r}_k^\alpha \rightarrow$  Solve the local contact problem;
     $\mathbf{v}_0^\alpha, \mathbf{v}_1^\alpha \rightarrow$  Apply the contact impulse  $\mathbf{r}_k^\alpha$ ;
    if  $\|\delta \mathbf{u}_k^\alpha - \delta \mathbf{u}_{k-1}^\alpha\| \Delta t < \xi$  then
      converged = True;
       $\mathbf{v}_0^\alpha, \mathbf{v}_1^\alpha \rightarrow$  Unapply the contact impulse  $\mathbf{r}_k^\alpha$ ;
       $\mathbf{v}_0^\alpha, \mathbf{v}_1^\alpha \rightarrow$  Apply the contact impulse  $\mathbf{r}_{k-1}^\alpha$ ;
    end
  end
end

```

Optimisation As observed in the previously described algorithm, even if only a few contacts require additional iterations to converge, the algorithm continues to loop over all contacts at each step. This inefficiency has been addressed by introducing a queue to manage contact resolution more effectively. Initially, all detected contacts are inserted into the queue. We then iterate through the queue, solving one contact at a time. If a contact fails to converge, it is reinserted back into the queue, along with all contacts involving the same bodies. This strategy focuses computational effort on unresolved contacts and their immediate neighbors, which are most likely to be affected. Consequently, the algorithm becomes more efficient by dynamically adapting its focus to the regions of the system where convergence has not yet been achieved.

Conclusion

In this chapter, we present the Non-Smooth Contact Dynamics (NSCD) framework for modelling contact interactions between rigid bodies. This method treats contact events as instantaneous and discontinuous, represented through impulses. The NSCD approach enables the robust simulation of systems with many simultaneous contacts, such as granular materials.

Algorithm 3 summarises the overall procedure for the NSCD method. The free velocity \mathbf{v}^- is computed based on the external forces acting on the bodies. Potential contacts are detected, and the contact impulse is computed using the Nonlinear Gauss-Seidel algorithm. Finally, the bodies positions are updated

based on the computed velocities. The time step Δt must be chosen to ensure stability in the explicit force computation and to allow for an accurate contact detection.

Algorithm 3: NSCD Overview.

Require: $\mathbf{q}^n, \mathbf{v}^n, \mathbf{f}_e^n, \Delta t$

\mathbf{v}^-	\leftarrow	$\mathbf{v}^n + \Delta t \mathbf{f}_e^n$
\mathcal{C}	\leftarrow	Contact Detection
$\mathbf{p}^{n+1}, \mathbf{v}^{n+1}$	\leftarrow	NonLinear Gauss-Seidel
\mathbf{q}^{n+1}	\leftarrow	Update the bodies position

In the next chapter, we turn our attention to the fluid phase representation and detail the coupling strategy between the fluid and solid phases.

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